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1 **Investigation of different nanoparticles for**
2 **magnetophoretically enabled nanofin heat sinks in**
3 **microfluidics**

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19 **Abstract**

20 Assembled nanofin heat sinks, nanostructures which are formed *via* external forces in a cooling
21 microfluidic to remove heat from hot spots, is a new concept that has recently been introduced.
22 In this work we investigate nanofin structures formed by CrO₂ and Fe₂O₃ magnetic nanoparticles
23 and compare their performance. Thermal imaging are used for comparison of the three cases
24 including: (i) DI water as the coolant liquid, (ii) suspension of magnetic particles in DI water,
25 and (iii) suspension of magnetic particles in DI water in the presence of a magnetic field. For
26 each case, the experiments are conducted at three different flow rates of 10, 40 and 120 $\mu\text{l min}^{-1}$.
27 Our results suggest that the high thermal conductivity of the nanofins comprising of CrO₂
28 significantly enhances the heat exchange across the microchannel. The proof-of-concept
29 magnetophoretic system can offer a practical solution for the cooling of future compact
30 electronics.

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32 **Keywords:** Cooling; heat exchange; hot spots; magnetophoresis; microfluidics; permanent
33 magnet

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42 **Introduction**

43 Ever more packing of transistors into integrated circuits (ICs), in order to achieve higher
44 processing powers, signifies the problem of cooling of hot spots, which are generated during the
45 operation of these electronic systems. Without any proper thermal management, the temperatures
46 of these hot spots significantly rise, thus reducing the expected life and reliability of the ICs.
47 Hence, effective cooling of such compact systems has become a major challenge facing the
48 development of future electronic components^{1,2}.

49 Cooling by means of liquid types such as water, ethylene glycol or various engine oils,
50 has been proposed to cool the hot spots in microchannels integrated with heat sinks^{3,4}. To date,
51 fin structure heat sinks with various geometries have been developed for cooling applications⁵⁻⁷.
52 Further improvement in heat transfer has been achieved by growing high aspect ratio microfins
53 made of highly thermally conductive materials (including carbon nanotubes)^{8,9}. However,
54 this approach is costly, not quite compatible with many silicon industry standards, time
55 consuming and involves rather complicated fabrication processes.

56 To increase the thermal conductivity of the coolant fluid, nanoparticles are added to the
57 liquid forming suspensions known as “nanofluids”¹⁰. Implementing nanofluids are suggested to
58 be an attractive solution for cooling micro-scale devices¹¹⁻¹³. Nevertheless, addition of
59 nanoparticles to the liquid has a limit, since high concentrations of nanoparticles increases the
60 viscosity of the fluid, causing a pressure drop and can even interrupt the flow passage by
61 clogging the microchannel¹⁴⁻¹⁶. To overcome such a problem, we can use other advantages that
62 exist from nanoparticle materials. Nanoparticles act as near free particles in liquid suspensions.
63 As a result, they can be manipulated to be aligned or trapped in specific locations of the
64 microchannel^{17,18}. By doing this, while the nanoparticles’ concentration can be increased at the

65 desired location, whilst their average concentration within the microfluidic system can be kept at
66 a low magnitude to allow for facile liquid flow^{19, 20}.

67 Among nanofluids are ferrofluids that consist of nano-scale magnetic particles in a non-
68 magnetic liquid, and have been widely used for cooling of microelectronic devices²¹⁻²⁴. In this
69 case, the suspended magnetic nanoparticles can be manipulated by external magnetic fields to
70 enhance heat transfer along the well aligned magnetic nanoparticles. Microelectromagnets can
71 produce high gradient magnetic fields in order to guide the target magnetic particles^{25, 26}.

72 We have previously shown that CrO₂ magnetic nanoparticles can be dynamically formed
73 onto the hot spots magnetophoretically to allow the efficient heat exchange with coolant liquid in
74 a microchannel¹⁹. We demonstrated the formation of bundles of micro-size and long fins from
75 the assembled nanoparticles, which we called nanofins, which could significantly enhance the
76 cooling process. This was due to the high aspect ratio of the fins and their flexible structure that
77 could allow the large interaction of the liquid coolant with them. This paper is an extension to
78 this previous work. We compare the performance of two different magnetophoretically
79 assembled nanoparticles: CrO₂ nanorods with high thermal conductivity as well as a long
80 morphological structure and Fe₂O₃ nanoparticles with a relatively lower thermal conductivity.
81 The thermal performance of the systems are assessed by measuring the temperature profiles
82 using an infrared camera. Extensive experimental and numerical analysis are conducted to
83 explore the effectiveness of these two model nanoparticles in forming nanofins at different flow
84 rates of the liquid.

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88 **Experimental section**

89 **Magnetic particles preparation**

90 CrO₂ and Fe₂O₃ nanoparticles, were purchased from Sigma-Aldrich, Australia, and their typical
91 morphologies are shown in Fig. 1(a and b), respectively. CrO₂ nanoparticles have a rod- shape of
92 35 nm diameter and 250 nm length on average, while Fe₂O₃ nanoparticles are spherical with the
93 average diameter of 184 nm as confirmed by Distribution Light Scattering (DLS) system (ALV-
94 GmbH, Germany) (see Fig. S3). Separate solutions of CrO₂- Mili-Q water and Fe₂O₃-Mili-Q
95 water were prepared as detailed below.

96 Both dispersions were stabilized with trisodium citrate dehydrate (Na₃C₆H₅O₇, 2H₂O) at
97 room temperature to avoid aggregation. To achieve this, trisodium citrate dehydrate 300 µl at
98 5 mg ml⁻¹ was added to Mili-Q water 15 ml, and then the nanoparticles were added. This solution
99 was sonicated for 30 min to reduce aggregates. Next, the solution was placed in a thermo-mixer
100 at 70 °C with a speed of 600 rpm for 8 hours to functionalize the surface of each type of
101 nanoparticles with citric acid. The solution was then centrifuged at 8000 rpm for 15 min and
102 washed with Milli-Q water. This process was repeated three times, and the particles were re-
103 dispersed in 15 ml Milli-Q water to a concentration of suspended nanoparticles of 0.15% w/w.
104 We further diluted each solution 2.5 times in DI water, which reduced the final concentration to
105 0.06% w/w (0.012% v/v). Transmission Electron Microscopy (TEM, Jeol 1010 TEM) and
106 Scanning Electron Microscopy (FEI Nova SEM 650) were conducted to verify the size and shape
107 of the CrO₂ and Fe₂O₃ nanoparticles, respectively.

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112 **Microfluidic device**

113 The magnetophoretic platform is schematically presented in Fig. 1(c). It consists of a
114 polydimethylsiloxane (PDMS) block, a heater and a glass slide. The microchannel dimensions
115 were set to $1500\ \mu\text{m} \times 280\ \mu\text{m} \times 40\ \text{mm}$ (width \times height \times length). The channel height of $280\ \mu\text{m}$
116 was chosen to prevent any possible clogging of the microchannel after the entrapment of
117 nanoparticles. The microchannel was fabricated from PDMS using soft photolithography
118 techniques²⁷. In doing so, SU8-3050 (Microchem, USA) layer was spin coated three times at
119 1000 rpm on a 3-inch diameter silicon wafer to produce a $280\ \mu\text{m}$ thickness layer. The sample
120 was then exposed to UV light source using an MA6 mask aligner for 3 min with an interval of
121 1 min between each exposure, and developed in SU-8 developer for 40 min to realize the
122 patterns on the master. A 10 g mixture of PDMS base and curing agent (Sylgard 184, Dow
123 Corning) were mixed in a 10:1 weight ratio, and degassed in order to remove the trapped air
124 bubbles using a vacuum oven. The PDMS mixture was poured onto the master, such that it
125 covered the microchannel and cured on a hot plate with a temperature of $70\ ^\circ\text{C}$ for 5 min.

126 Next, a small heater made by winding a Eureka wire ($2.2\ \Omega/\text{foot}$) around a 5 mm diameter
127 ceramic ring, was integrated onto the PDMS block. In order for heat to easily travel through the
128 microchannel, the heater was inserted 1 mm from a microchannel side wall. The rest of the
129 PDMS mixture was poured into the device to cover the heater. The device was cured for a further
130 15 min on the 70°C hot plate. The substrate was allowed to cool down for 5 min. The PDMS
131 block of $50\ \text{mm} \times 10\ \text{mm} \times 5\ \text{mm}$ (length \times width \times height) was carefully peeled from the master.
132 The fabrication process was conducted in a class 1000 cleanroom.

133 The PDMS block was integrated onto a glass slide (Menzel-Glaser, USA) of $60\ \text{mm} \times 20$
134 $\text{mm} \times 100\ \mu\text{m}$ (length \times width \times height). The glass thickness of $100\ \mu\text{m}$ was chosen to facilitate the
135 temperature measurement using infrared camera. Tygon® microtubes with an internal diameter

136 of 400 μm were placed into the holes punched within the PDMS block to interface with the
137 sample bottle and the syringe pump.

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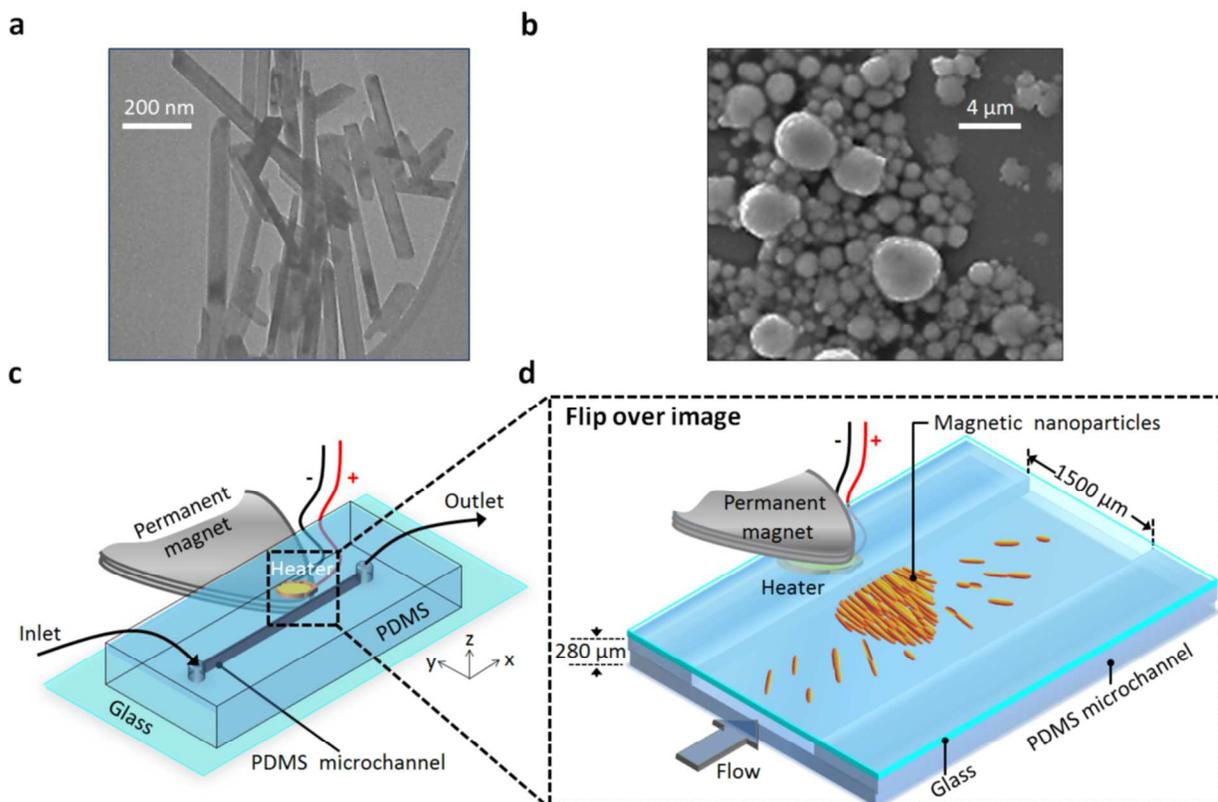
139 **Permanent magnet**

140 Three sheets of nickel plated neodymium magnet, which are used in standard hard disks of
141 computers, with dimension of 40 mm \times 20 mm \times 2 mm (length \times width \times height), were stacked
142 on top of each other to produce a total uniform magnetic field of ~ 0.36 T at the microchannel
143 region, as measured by a Teslameter (F. W. BELL, USA). The magnet stack was oriented in
144 such a way that its tip was exposed to the hot spot (Fig. 1(d)). The magnetic field drops with the
145 distance from the microchannel was measured and showed in Fig. S4.

146

147 **Apparatus**

148 Supplementary Fig. S1 shows an experimental setup of the microfluidic system. A syringe pump
149 (Harvard PHD 2000) was used for providing the flow through the microchannel. The syringe
150 pump was activated in refill mode to supply a suction force in order to prevent the leakage and
151 generation of bubbles within the microchannel. The heater was energized *via* a DC power supply
152 (Gw Instek, GPS-X303 series, Taiwan). Before each experiment, to prevent the adhesion of
153 $\text{CrO}_2/\text{Fe}_2\text{O}_3$ particles to the glass and PDMS surface, the channel was flushed with a 1% *w/w*
154 mixture of liquid surfactant (Triton X-305) for 10 min at a flow rate of $80 \mu\text{l min}^{-1}$ and then
155 washed with DI water for 10 min. The microfluidic device was mounted on an inverted
156 microscope (Nikon, Eclipse TE2000U, Japan) to observe the trapping of nanoparticles in time.
157 An infrared camera (Titanium Cedip Infrared Systems, France) was used for measuring the
158 temperature along the glass slide, which formed the bottom surface of the microchannel.



159

160 **Fig. 1** The microfluidic platform and magnetic nanoparticles. (a) TEM image of CrO_2
161 nanoparticles. (b) SEM image of Fe_2O_3 nanoparticles. (c) The schematics of microfluidic system
162 comprised of a PDMS block bonded to a 100 μm glass slide, an integrated heater, and a
163 permanent magnet. (d) The close up of the microchannel bottom view with trapped CrO_2
164 magnetic nanoparticles.

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172 Simulation details

173 Extensive computational fluid dynamic (CFD) simulations were carried out to further analyse the
 174 thermal performance of the magnetophoretic system. The simulations were conducted in both
 175 three-dimensions and steady state using the ANSYS Fluent 6.3 (Canonsburg, PA, USA) software
 176 package. This involved solving the differential equations governing the balance of mass,
 177 momentum and energy to calculate the variations of velocity, pressure and temperature
 178 throughout the microchannel, as given below:

$$\nabla \cdot \vec{U} = 0 \quad (1)$$

$$\rho_{suspension} (\vec{U} \cdot \nabla) \vec{U} = -\nabla P + \mu_{suspension} \nabla^2 \vec{U} \quad (2)$$

$$(\rho \cdot c_p)_{suspension} (\vec{U} \cdot \nabla) T = k_{suspension} \nabla^2 T \quad (3)$$

179 where \vec{U} , P and T are the velocity, pressure and temperature of the suspension, ρ , μ , c_p and k are
 180 the density, dynamic viscosity, heat capacity and thermal conductivity of the suspension. The
 181 values of ρ , μ , c_p and k for the DI water, CrO_2 suspension, and Fe_2O_3 suspension are given, as
 182 presented in the Supplementary Information 2.

183 The simulations involved solving the energy equation for the heater, PDMS block,
 184 magnetically formed nanofins, and glass slide, as given below:

$$k_{heater} \nabla^2 T + \dot{Q}_{heater} = 0 \quad (4)$$

$$k_{PDMS} \nabla^2 T = 0 \quad (5)$$

$$k_{nanofin} \nabla^2 T = 0 \quad (6)$$

$$k_{glass} \nabla^2 T = 0 \quad (7)$$

185 where k_{heater} , k_{PDMS} , $k_{nanofin}$ and k_{glass} are the thermal conductivities of the heater, PDMS block,
186 nanofins and glass slide, respectively, while \dot{Q}_{heater} is the source term representing the power
187 injected *via* the heater.

188 The boundary conditions used for calculating the variations of velocity and pressure
189 within the microchannel include the zero pressure at the inlet and the desired flow rate at the
190 outlet of the microchannel. No slip boundary condition was applied at the surfaces of the channel
191 for the pure liquid, which was replaced with the slip boundary condition for the CrO₂ or Fe₂O₃
192 suspensions, as detailed in our previous work¹. The boundary conditions used for calculating the
193 variations of temperature within the solid parts included: the ambient temperature at the inlet of
194 the channel, a fully developed boundary condition at the outlet of the channel, and free
195 convection at the free surfaces of the system in contact with the surrounding environment with
196 the natural convection coefficient assumed to be 10 W m⁻² K⁻¹. The value of \dot{Q}_{heater} was
197 calculated by considering the injected power as 0.2 W.

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208 **Results**

209 **Trapping characteristics of nanoparticles**

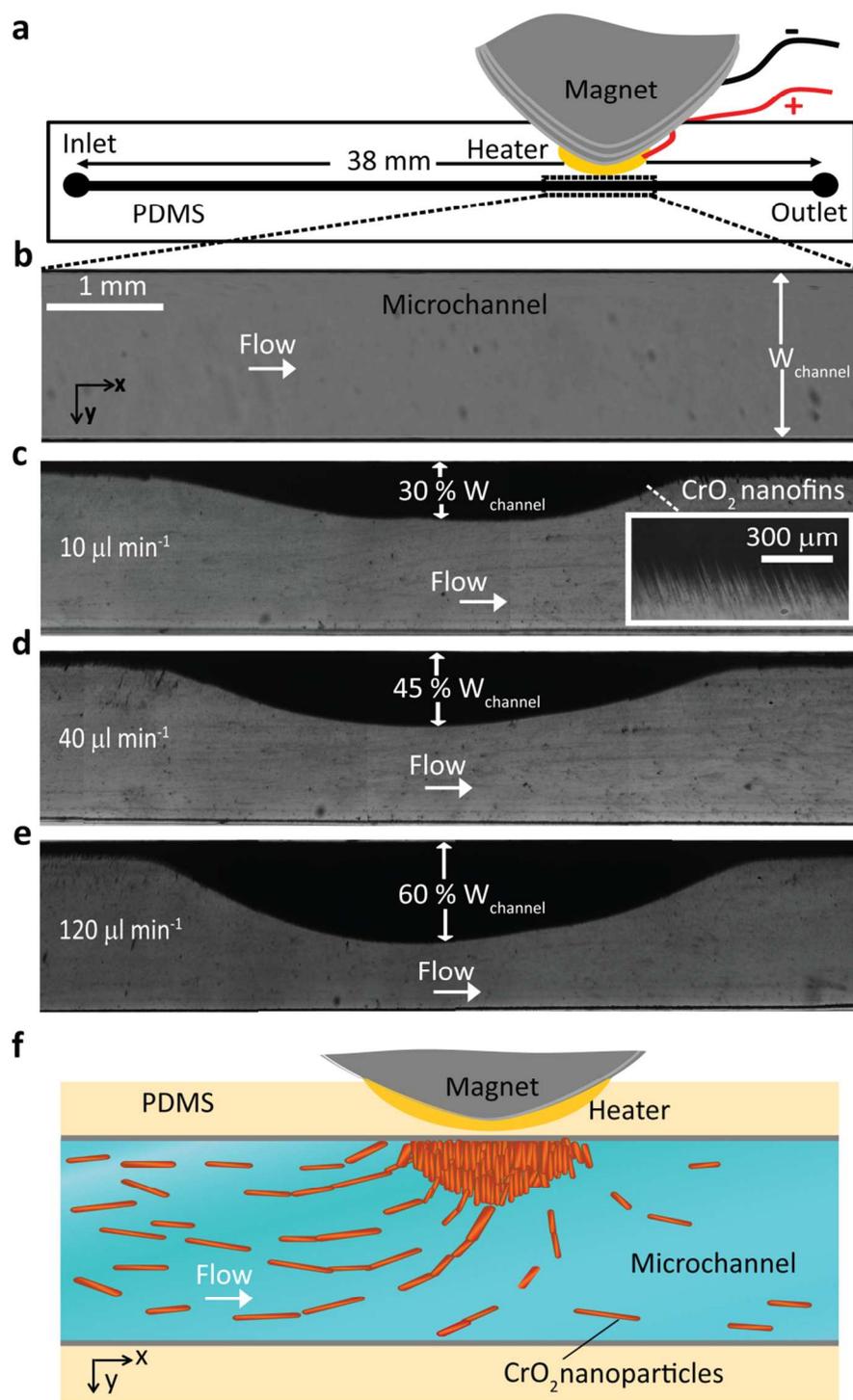
210 The trapping mechanism of CrO_2 and Fe_2O_3 nanoparticles was studied at the flow rates of 10, 40
211 and $120 \mu\text{l min}^{-1}$. The permanent magnet was placed at the bottom of the device (Fig. 1(d)) to
212 provide a strong magnetic field within the microchannel. To minimize the heat conduction
213 between the hot glass slide and the surface of the permanent magnet, the magnet was isolated
214 from the glass by a 0.5 mm layer of rubber (Blu-Tack). Moreover, to ensure that the hot region
215 surrounding the heater is visible by the infrared camera, the tip of the magnet was placed 1.5 mm
216 from the microchannel (Fig. 2(a)). The heater was energized with a constant DC current of 0.5 A
217 and voltage of 0.4 V for 10 min to allow the system reach its steady state temperature before
218 applying the magnet.

219 Fig. 2(a) shows a schematic represents the microchannel. Fig. 2(b) demonstrates a flow of
220 DI water within the microchannel. Fig. 2(c-e) depict the CrO_2 nanoparticles trapped along the
221 side wall of the microchannel 10 min after the application of magnetic field (refer to
222 Supplementary Fig. S5 for Fe_2O_3 nanoparticles trapping under various flow rates). The images
223 were captured by a 4× microscope objective lens and assembled along the x -axis to demonstrate
224 the full picture.

225 Approaching the magnet, the nanoparticles began to form chains, were deflected towards
226 the side wall (near hot spot), and also docked perpendicular to the side wall (as demonstrated in
227 Fig. 2(f). The docked chains grew to form micro-sized bundles of CrO_2 nanoparticles (nanofins)
228 along the side wall with maximum lengths closer to the tip of the magnet (Fig. 2(f)).

229 At a low flow rate of $10 \mu\text{l min}^{-1}$, the maximum width of nanofins reached $450 \mu\text{m}$, or
230 30% of the microchannel width ($0.3W_{\text{channel}}$). At the same time, the total length of the nanofin

231 arrays reached $\sim 4600 \mu\text{m}$ (i.e. $\sim 3.1 W_{\text{channel}}$). Interestingly, the maximum width and length of the
232 nanofins increased with an increase in flow rate Fig. 2(d and e). For example, at the flow rate of
233 $40 \mu\text{l min}^{-1}$, they increased to $\sim 0.45 W_{\text{channel}}$ and $\sim 3.5 W_{\text{channel}}$, respectively. Likewise, at a high
234 flow rate of $120 \mu\text{l min}^{-1}$, they reached $\sim 0.68 W_{\text{channel}}$ and $\sim 3.9 W_{\text{channel}}$, respectively (see
235 Supplementary video-1 and 2 for the formation of CrO_2 and Fe_2O_3 nanofins at the above
236 conditions, respectively). The system had a saturation time, after which the growth of nanofins
237 was decelerated. Under the aforementioned conditions, the saturation was observed 8-10 min
238 after the application of the magnetic field. The nanofins can be washed away from the side wall
239 by removing the magnet and applying DI water to the microchannel at a flow rate of $200 \mu\text{l min}^{-1}$
240 for 3 min. A full discussion on the dynamics of the nanofins can be found in our previous
241 work¹⁹.



242

243 **Fig. 2** (a) A schematic of the microchannel. Growing length of CrO_2 nanoparticles into nanofins244 under the influence of the magnetic field at different flow rates of 10 , 40 , and $120 \mu\text{l min}^{-1}$ after245 10 min. (b) A flow of DI water in the microchannel. (c) At $10 \mu\text{l min}^{-1}$ the length of CrO_2

246 nanofins were obtained to be 30% of the microchannel's width. The inset demonstrates a close
247 up image of the nanofins layer. (d) At $40 \mu\text{l min}^{-1}$ the length of CrO_2 nanofins were obtained to
248 be 45% of the microchannel's width. (e) At $120 \mu\text{l min}^{-1}$ the length of CrO_2 nanofins were
249 obtained to be 60% of the microchannel's width. (f) the schematic representing the growth of
250 nanofins.

251

252 **Thermal characterization of nanofins**

253 Next, we characterized the thermal performance of the magnetophoretic platform by obtaining
254 thermal images from the side of the $100 \mu\text{m}$ thick glass substrate using an infrared camera, as
255 described in our previous work¹. The measurements were conducted at three different flow rates
256 of 10, 40 and $120 \mu\text{l min}^{-1}$. The temperature of the fluid entering the microchannel followed the
257 ambient temperature, which was $298 \pm 0.3 \text{ K}$ during the experiments. The heater was energized by
258 applying a DC signal of 0.5 A and 0.4 V. The results obtained at 40 and $120 \mu\text{l min}^{-1}$ are shown
259 here and the case involving a $10 \mu\text{l min}^{-1}$ flow rate is presented in the Supplementary Fig. S6.

260 The thermal imaging was conducted for the cases of (i) DI water, (ii) CrO_2 suspension
261 without permanent magnet, and (iii) CrO_2 suspension with permanent magnet (Fe_2O_3 suspension
262 thermal images are shown in Supplementary Fig. S7). For the case (iii), we used a permanent
263 magnet close to glass slide, as described in the previous section. However, for the cases (i) and
264 (ii) there was no magnetic field present. In order to mimic the same situations for these two cases
265 and capture comparable geometries by the infrared camera, we placed a 1 mm thick aluminum
266 plate exactly at the location of the permanent magnet (see Fig. 1(d)). The aluminum plate was
267 separated from the glass slide by a thin layer of rubber (Blu-Tack) to insulate it thermally. In
268 comparison to CrO_2 , almost no change was discernible in thermal images of Fe_2O_3 suspensions

269 at similar cases. As a result, only the thermal profiles of Fe_2O_3 suspensions are shown in the next
270 section for comparison and discussion.

271 Fig. 3(a) shows a representation of the obtained temperature contours. Fig. 3(b-d) show
272 the obtained temperature contours at the glass surface for the aforementioned three cases at the
273 flow rate of $40 \mu\text{l min}^{-1}$. The results were captured after 20 min, with 10 min to ensure the
274 system had reached a steady state temperature with the flow of DI water, and an extra 10 min to
275 ensure nanofin structures had been formed whilst subjected to the magnetic field with the
276 constant flow of nanoparticles. The heat, generated by the heater, was propagated through the
277 PDMS *via* conduction, as evidenced by the symmetric distribution of temperature contours
278 across the PDMS. However after reaching the microchannel, there was a competition between
279 the conduction and convection mechanisms to convey the heat.

280 In case (i), the hot side wall of the microchannel reached a maximum temperature of
281 313.8 K (Fig. 3(b)). At a high flow rate of $40 \mu\text{l min}^{-1}$, the low thermal conductivity of DI water
282 led to a poor conduction through the microchannel, as evidenced by the stretching of temperature
283 contours towards the microchannel outlet (Fig. 3(b)). This led to a maximum temperature of
284 307 K at the opposite side of the microchannel and an average temperature of 304 K at the outlet
285 (Fig. 3(b)).

286 In case (ii), the heat exchange between the hot spot and the coolant fluid was improved
287 by adding CrO_2 nanoparticles (0.06% w/w) into the suspension. No magnetic field was applied in
288 this case. CrO_2 nanoparticles had a higher thermal conductivity than DI water, and increased the
289 overall thermal conductivity of the suspension, as evidenced by advancing the red contours
290 towards the opposite side wall and the outlet. This increased the maximum temperatures of the
291 hot and cold side wall to 315 K, and 308 K, respectively, while the average outlet temperature

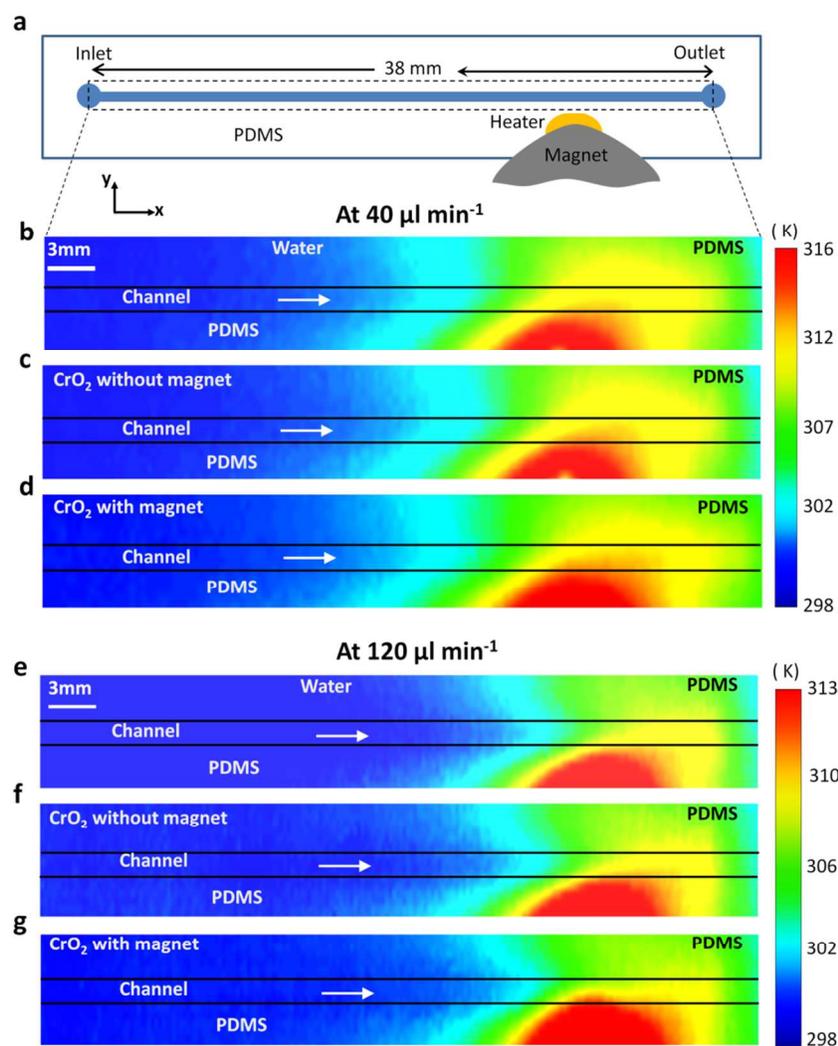
292 increased to 304.6 K microchannel (Fig. 3(c)). Similar effects have been demonstrated by other
293 researchers²⁸⁻³¹.

294 In case (iii), a permanent magnet was applied to the system. The magnet generated a
295 magnetic field and caused CrO₂ nanoparticle to form chains which eventually created nanofins at
296 the hot spot, as shown in Fig. 2(c). Given the high thermal conductivity of nanofins
297 ($31 \text{ W m}^{-1} \text{ K}^{-1}$)³², they provided an efficient path to conduct the heat from the hot side wall
298 towards the core of the microchannel, as evidenced by further advancing of red contours towards
299 the opposite side wall and the outlet (Fig. 3(d)). This increased the maximum temperature along
300 the hot and cold side walls to 316 K and 308 K, respectively while the average outlet temperature
301 increased to 306.5 K.

302 Fig. 3(e-g) show the obtained temperature contours at the glass surface for the three cases
303 at the flow rate of $120 \mu\text{l min}^{-1}$. Likewise, these results were captured 10 min after the system
304 had reached its steady state temperature. Similar trends were observed in the temperature
305 contours before and after the addition of nanoparticles and also after applying the magnetic field.
306 However, increasing the flow rate to $120 \mu\text{l min}^{-1}$ enhanced the convective heat transfer across
307 the microchannel, which resulted in a general temperature drop across the microfluidic platform
308 (Fig. 3(e-g)).

309 Simulations are also conducted to verify and further discuss the performance of the
310 systems with and without nanoparticles as well as in conditions in which the CrO₂ nanoparticles
311 are trapped in the presence of a magnetic field, and form nanofins. The optimum flow rate of
312 $120 \mu\text{l min}^{-1}$ is used for these simulations. Additionally, in order to compare the results, and
313 assess the conditions for increasing the thermal exchanges between the hot spot and the liquid
314 coolant in the channel, a set of thermal performance analyses was conducted by investigating the

315 temperature profiles along the parallel lines to the side walls, which are presented in the next
 316 section.



317
 318 **Fig. 3** (a) Schematics of the microchannel. Contours of temperature along the glass slide,
 319 obtained by infrared camera at flow rates of 40 and 120 $\mu\text{l min}^{-1}$ for the cases of: (b) Water
 320 flowing through the microchannel. (c) CrO₂ nanoparticle suspension in the absence of the
 321 magnet. (d) CrO₂ nanoparticle suspension in the presence of the magnet, forming CrO₂ nanofins
 322 along the side wall. (e) Water flowing through the microchannel. (f) CrO₂ nanoparticle

323 suspension in the absence of the magnet. (g) CrO₂ nanoparticle suspension in the presence of the
324 magnet, forming nanofins.

325

326 **Numerical simulation results**

327 Simulations were conducted at a flow rate of 120 $\mu\text{l min}^{-1}$ and in three different cases to be in
328 line with the experiments: (i) DI water flowing through the channel, (ii) a suspension of CrO₂
329 nanoparticles flowing through the channel, and (iii) a suspension of CrO₂ nanoparticles flowing
330 through the channel while a CrO₂ nanofin structure is formed along the hot side wall.

331 Fig. 4 shows the simulated temperature contours along the external surface of the glass
332 slide (which was experimentally measured using the thermal imaging camera) as well as along
333 the plane normal to the microchannel, which passes through the heater and magnetically formed
334 nanofins (which cannot be measured using the thermal imaging camera).

335 For case (i), the simulations indicated the propagation of heat towards the microchannel,
336 which was associated with the deflection of temperature contours towards the right (Fig. 4(a)),
337 which was also confirmed in the thermal imaging experiments. The cross sectional contours
338 indicated that the temperature does not significantly change along the thickness of the glass slide
339 and the height of channel. This means that the temperature contours obtained by thermal imaging
340 camera could correctly obtain the temperature variations across the channel (Fig. 4(b)).
341 However, a sharp temperature drop of 6.1 K was obtained across the width of the channel,
342 indicating the dominance of convection (Fig. 4(b)).

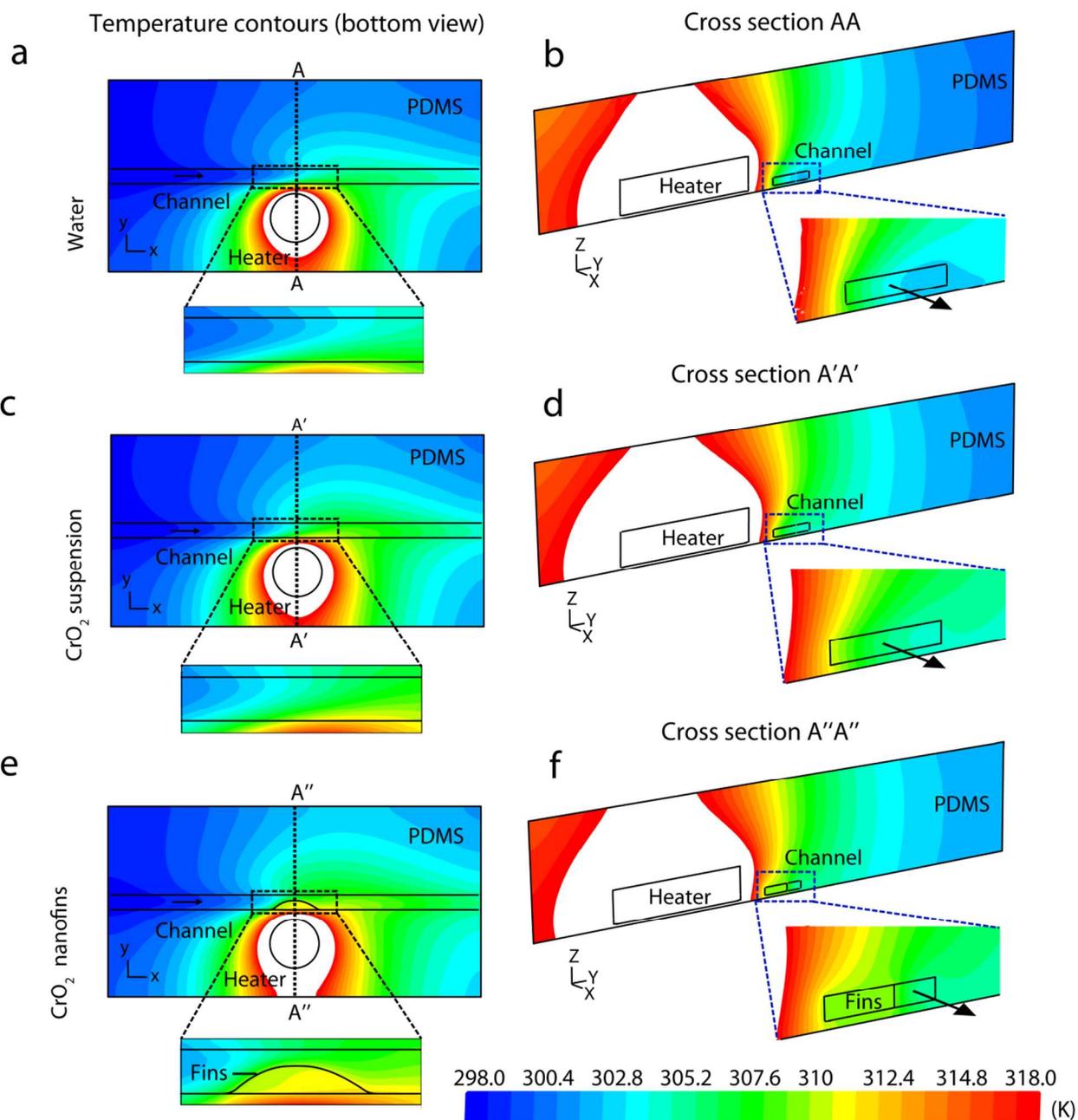
343 For case (ii), the heat was more smoothly propagated within the system, as evidenced by
344 advancement of 'yellow' and 'green' bands towards the microchannel, and also the receding of
345 the 'blue' band along the channel (Fig. 4(c)). Moreover, the temperature drop across the width of

346 the channel reduced to 5.05 K, indicating the improvement of the thermal conduction through the
347 channel (Fig. 4(d)).

348 For case (iii), the nanofin bundles were simulated using a structure (the conditions of this
349 structure are presented in our previous work¹⁹ formed adjacent to the hot side wall of the
350 microchannel with a geometry similar to that obtained in experiment (Fig. 2(e)), which extended
351 along the height of the channel. The simulations revealed that the formation of the CrO₂ nanofins
352 led to better propagation of heat through the entire microfluidic system, as evidenced by the
353 formation of a ‘yellow’ region at the location of nanofins, as also the receding of the ‘blue’ band
354 along the channel (Fig. 4(e)). More interestingly, the temperature drop across the width of the
355 channel reduced to 3.15 K (Fig. 4(f)). In other words, the incorporation of CrO₂ nanofins resulted
356 in an increase of a forced convection through the microchannel and also the free convection
357 across the free surfaces of PDMS block and the glass slide, which in turn avoided the
358 accumulation of heat at the hot spot.

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362 **Fig. 4** Simulation results for heat transfer within the magnetophoretic system. The flow rate of

363 the outlet is set to $120 \mu\text{l min}^{-1}$ while the temperature of the liquid at the inlet is set to 298 K.

364 Temperature contours across the glass slide and cross section of the microchannel for the case of:

365 (a-b) water. (c-d) CrO₂ nanoparticles suspension in the absence of magnet and (e-f) magnetically

366 assembled CrO₂ nanofins.

367 **Thermal performance of CrO₂ vs. Fe₂O₃ nanofins**

368 To further explore the effectiveness of CrO₂ and Fe₂O₃ nanofin structures, we studied the
369 thermal performance of the system under different flow rates of 10, 40 and 120 $\mu\text{l min}^{-1}$. In all
370 cases, the heater was energized by applying a DC signal of 0.5 A and 0.4 V. Both nanoparticle
371 suspensions were applied under the same concentration (0.06% w/w), and the results were
372 obtained after 10 min. The variations of temperature were examined along nine parallel lines
373 across the channel including both side walls, thus there was a gap of 187.5 μm gap between the
374 lines. At each flow rate, temperature variations were studied for the cases of (i) DI water, (ii)
375 CrO₂ nanoparticle suspension without magnet, (iii) CrO₂ nanoparticle suspension with magnet,
376 (iv) Fe₂O₃ nanoparticle suspension without magnet, and (v) Fe₂O₃ nanoparticle suspension with
377 magnet. When comparing different cases, we focused on three characteristic temperatures of the
378 system, including: the maximum temperature of the channel side wall close to the heater, T_{hot} ,
379 the maximum temperature of the channel side wall far from the heater, T_{cold} , and the average
380 temperature of the outlet, T_{outlet} . Here, we demonstrate the thermal performance of the system at
381 the flow rates of 40 and 120 $\mu\text{l min}^{-1}$, while the results obtained at a low flow rate of 10 $\mu\text{l min}^{-1}$
382 are given in Supplementary Fig. S9. For the case involving the constant flow rate of 10 $\mu\text{l min}^{-1}$,
383 the conduction was more dominant than the convection so it could not be directly compared to
384 40 and 120 $\mu\text{l min}^{-1}$ cases.

385 At a low flow rate of 40 $\mu\text{l min}^{-1}$, application of DI water case (i), led to the following
386 temperatures along the microchannel: $T_{hot} = 313.5$ K, $T_{cold} = 307$ K and $T_{outlet} = 304$ K (Fig. 5(a)).
387 Alternatively, addition of CrO₂ nanoparticles to DI water case (ii), led to a slight increase of
388 temperatures along the microchannel: $T_{hot} = 315$ K, $T_{cold} = 308$ K and $T_{outlet} = 304.6$ K (Fig. 5(b)),
389 which can be attributed to higher thermal conductivity of the suspension. Interestingly, the

390 activation of the magnetic field in the presence of CrO₂ nanoparticle suspension case (iii), led to
391 further increase of temperatures along the microchannel: $T_{hot} = 316$ K, $T_{cold} = 308$ K and $T_{outlet} =$
392 306.5 K (Fig. 5(c)), which can be attributed to strong conduction along the patterned nanofin
393 structure. The extent of the formed nanofin is shown in the graph. Moreover, a temperature drop
394 of 4 K was observed from the base to the tip of the nanofin.

395 Increasing the flow rate to $120 \mu\text{l min}^{-1}$, enhanced the convective heat transfer through
396 the microchannel, and thus reduced the overall temperature across the system. For the case of DI
397 water, case (i), the following temperatures were measured along the microchannel: $T_{hot} = 309$ K,
398 $T_{cold} = 304$ K and $T_{outlet} = 303.3$ K (Fig. 5(a)). Addition of CrO₂ nanoparticles to DI water,
399 case (ii), slightly increased the temperatures along the microchannel: $T_{hot} = 310$ K, $T_{cold} = 304$ K
400 and $T_{outlet} = 304$ K (Fig. 5(b)), as observed before. Activation of magnetic field in the presence of
401 CrO₂ nanoparticle suspension, case (iii), further increased the temperature along the
402 microchannel: $T_{hot} = 313$ K, $T_{cold} = 304$ K and $T_{outlet} = 304.2$ K (Fig. 5(c)). Moreover, the
403 temperature drop from the base to the tip of the nanofin increased to 6.4 K.

404 In contrast, the addition of Fe₂O₃ nanoparticles to DI water, case (iv), led to almost
405 similar trends observed for the DI water at both flow rates of 40 and $120 \mu\text{l min}^{-1}$ (Fig. 5(d)).
406 However, activation of magnetic field in the presence of Fe₂O₃ nanoparticle suspension, case (v),
407 increased the overall temperature along the microchannel: $T_{hot} = 314$ K, $T_{cold} = 306.2$ K and
408 $T_{outlet} = 304.2$ K at a flow rate of $40 \mu\text{l min}^{-1}$ (Fig. 5(e)). Moreover, a temperature drop of 4.5 K
409 was observed from the base to the tip of the Fe₂O₃ nanofin. Compared to the case of CrO₂
410 nanoparticle suspension, case (iii), temperature increase was less, which can be attributed to the
411 low thermal conductivity of Fe₂O₃ nanoparticles ($6 \text{ W m}^{-1} \text{ K}^{-1}$)³³ compared to CrO₂ nanoparticles
412 ($31 \text{ W m}^{-1} \text{ K}^{-1}$)³². Moreover, the spherical structure of Fe₂O₃ nanoparticles led to more phonon

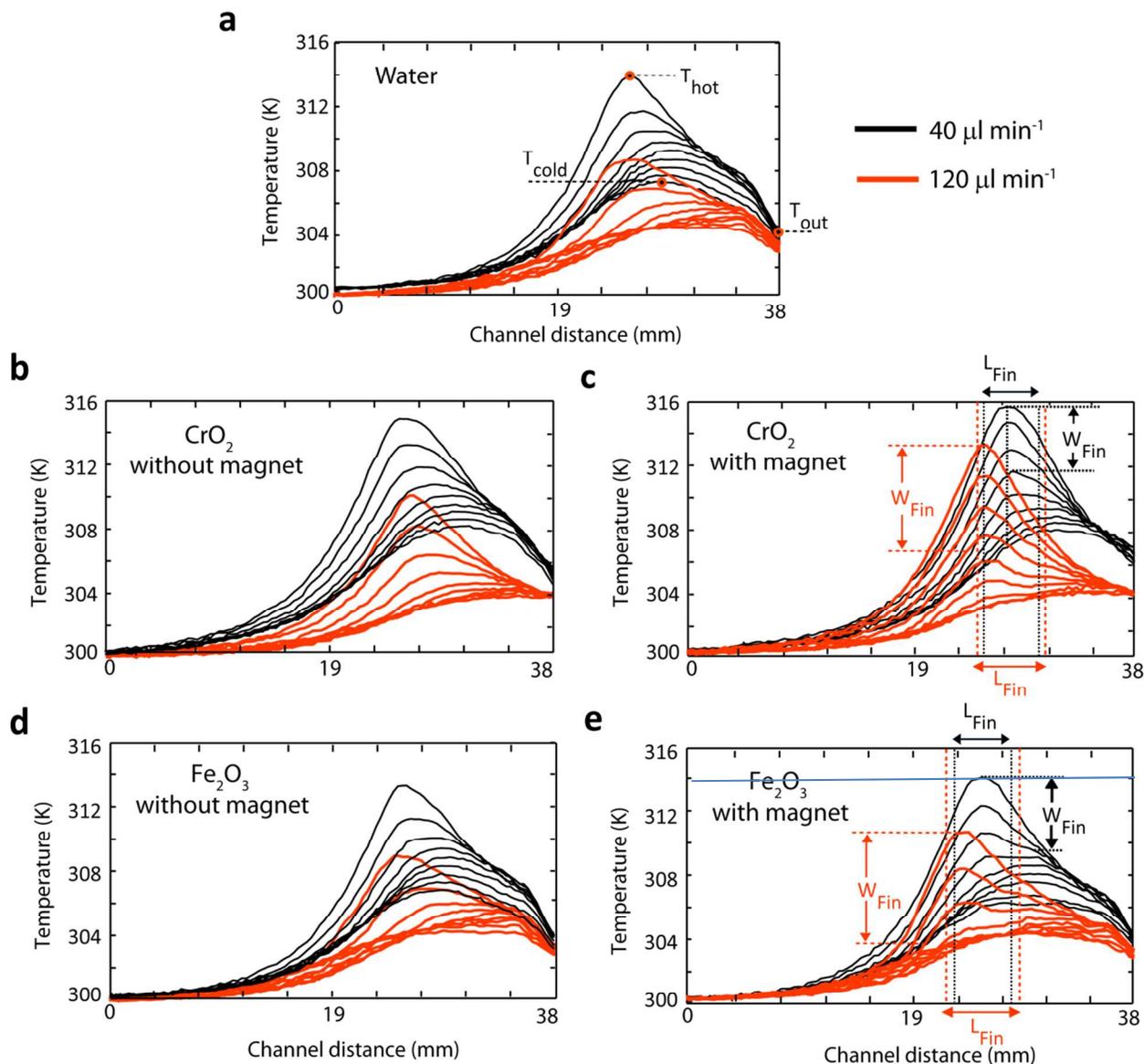
413 scattering, which in turn could cause a heat loss along the chains. In contrast, the CrO_2
414 nanoparticles had a rod-shape structure, as shown in SEM images (Fig. 1(a)), allowing phonons
415 to propagate with minimal scattering, which imposed less heat loss along the chains. Increasing
416 the flow rate to $120 \mu\text{l min}^{-1}$ led to similar trends and the following temperatures were measured
417 along the microchannel: $T_{hot} = 311 \text{ K}$, $T_{cold} = 304.2 \text{ K}$ and $T_{outlet} = 303.5 \text{ K}$ at a flow rate of
418 $40 \mu\text{l min}^{-1}$ (Fig. 5(e)). The temperature drop from the base to the tip of the Fe_2O_3 nanofin
419 increased to 7 K.

420 A set of experiments were conducted to observe the formation of the nanofin bundles at
421 different widths of the microfluidic channel, which is presented in Supplementary Information 9.
422 A study on the effect of various morphologies (aspect ratios) of CrO_2 and Fe_2O_3 on thermal
423 conductivity of nanofins is shown in Supplementary Information 3. The results indicate that an
424 elongated morphology such as rod-shaped (CrO_2) performs better for heat conduction
425 (Supplementary Fig. S2).

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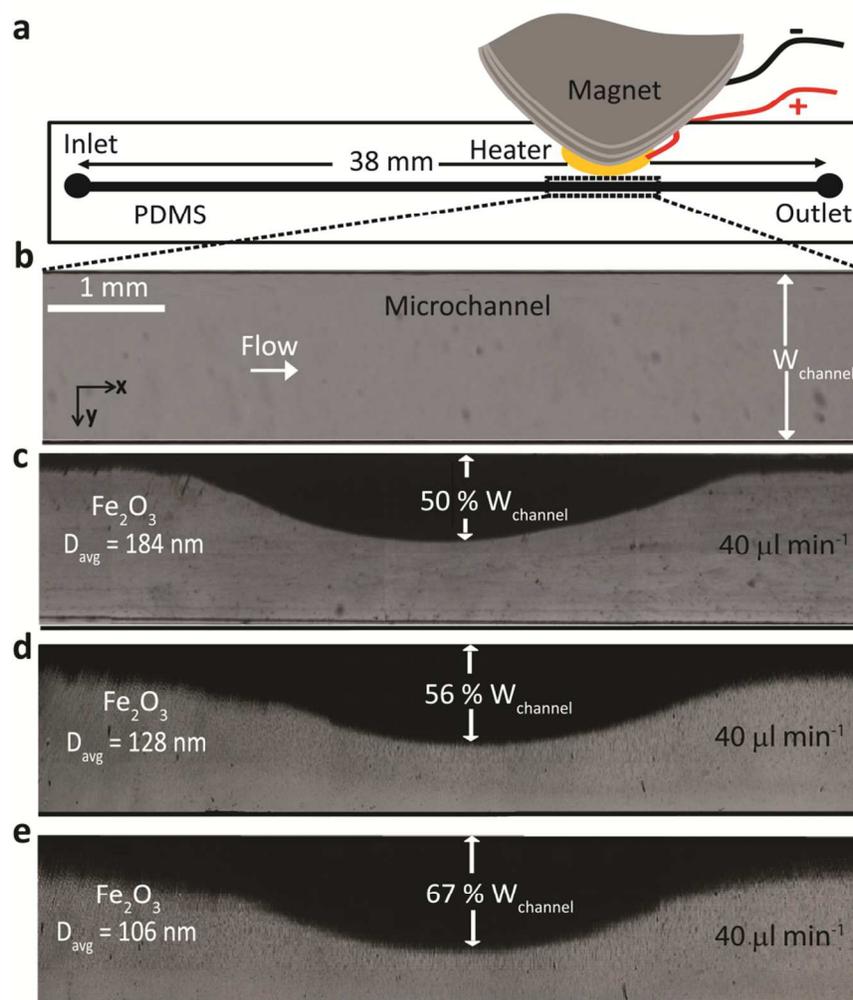
430 **Fig. 5** Thermal effect of Fe_2O_3 and CrO_2 nanofins along the microchannel at flow rates of 40 and431 $120 \mu\text{l min}^{-1}$: (a) DI water, (b) CrO_2 nanoparticles suspension without applying the magnet, (c)432 CrO_2 nanofins after applying the magnet, (d) Fe_2O_3 nanoparticles suspension without applying433 the magnet, and (e) Fe_2O_3 nanofins after applying the magnet.

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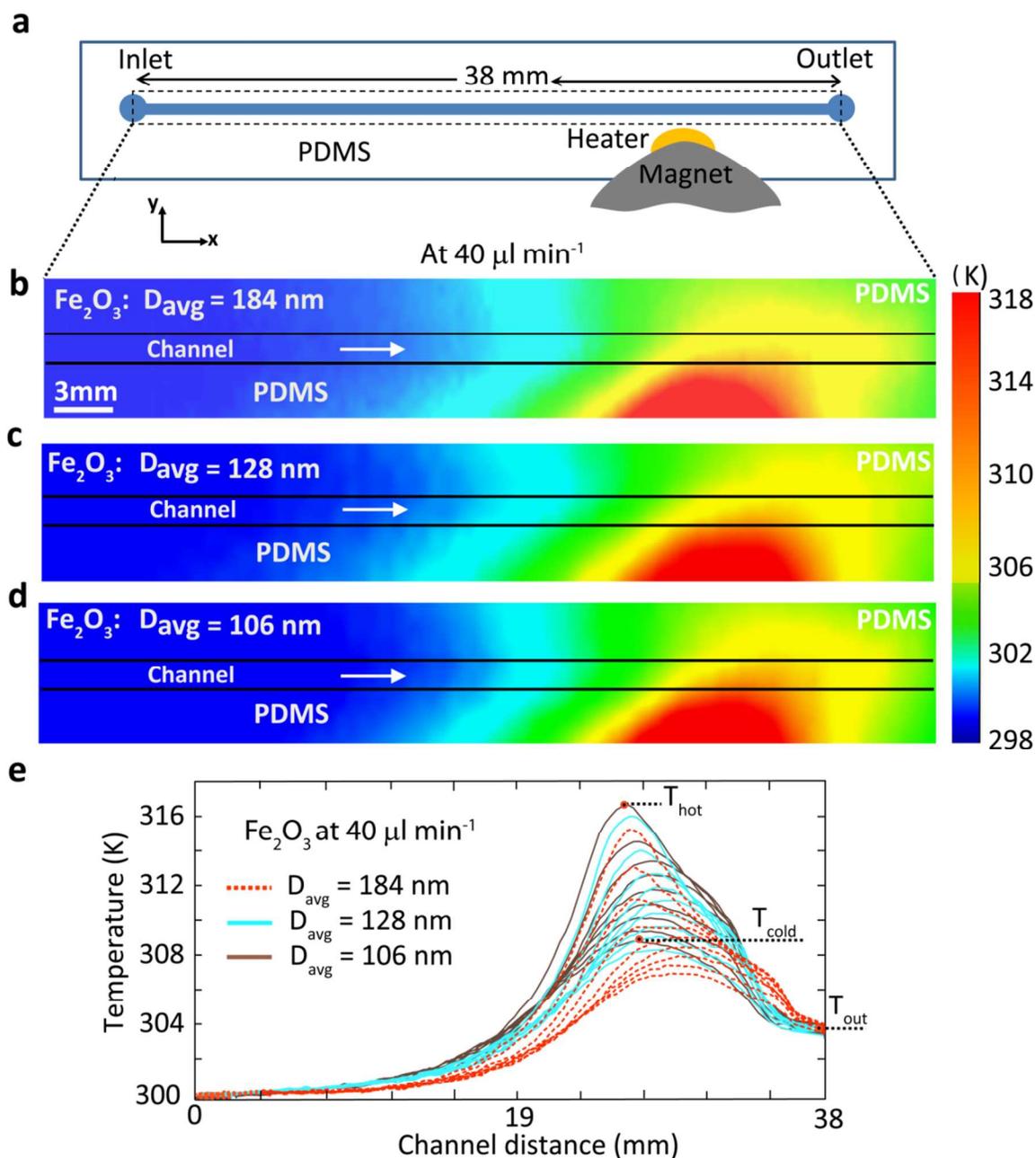
436 In the last stage of the experiments, the study of different size Fe_2O_3 nanoparticles in
437 liquid was conducted to observe the effect of the size of the particles forming the nanofin
438 bundles on the heat transfer. The particles were separated using filters of different sizes. As a
439 result, three different suspension solutions were obtained for comparison. DLS was used for
440 measuring the average diameter of the particles suspended in DI water for three different samples
441 (see Supplementary Information 4): (1) no filter with the average diameter of 184 nm, (2) filter
442 with the porosity of 450 nm that resulted in the average diameter 128 nm and (3) 220 nm porous
443 to obtain particles of average diameter of 106 nm.

444 The experiments were carried out at the flow rate of $40 \mu\text{l min}^{-1}$. The results are shown in
445 Figs. 6 and 7. As can be seen in Fig. 6, after 10 min the nanofins height reached 50%, 56% and
446 67% of the channel width for suspensions of 184 nm, 128 nm and 106 nm particles, respectively.
447 As can be seen in Fig. 7, the nanofins formed using the smaller particles (106 nm) shows an
448 improvement in heat transfer and the maximum temperature (T_{cold}) at the opposite side wall of
449 the microchannel was increased by ~ 2 K compared to the nanofins formed by the larger
450 nanoparticles (184 nm). This better heat transfer for smaller particles can be attributed to two
451 reasons. Longer nanofins are formed that can more efficiently transfer the heat into and other
452 side of the microchannel as well as the fact that more nanoparticles are in intimate contact with
453 each other, which promoted the thermal conductivity.



454
 455 **Fig. 6** (a) A schematic of the microchannel. (b) A flow of DI water representing the trapped
 456 section of nanoparticles. Growing length of Fe₂O₃ nanoparticle bundles (nanofins) under the
 457 influence of the magnetic field at a flow rate of $40 \mu\text{l min}^{-1}$ after 10 min for the cases of: (c) With
 458 184 nm average sized Fe₂O₃ nanoparticles, the length of Fe₂O₃ fins was obtained to be 50% of
 459 the microchannel's width. (d) With 128 nm average sized Fe₂O₃ nanoparticles, the length of
 460 Fe₂O₃ fins was obtained to be 56% of the microchannel's width. (e) With 106 nm average sized
 461 Fe₂O₃ nanoparticles, the length of Fe₂O₃ fins reached 67% of the microchannel's width.

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464 **Fig. 7 (a)** Schematics of microchannel. Temperature contours along the glass slide obtained
 465 using infrared camera at flow rates of $40 \mu\text{l min}^{-1}$ for the cases with average diameters of: (b)

466 184 nm, (c) 128 nm, and (d) 106 nm Fe_2O_3 nanoparticles, respectively. (e) Temperature

467 distribution for different size of Fe_2O_3 nanoparticles forming nanofins along the microchannel at

468 flow rates of $40 \mu\text{l min}^{-1}$ (the temperature profiles are taken similar to those of Fig. 5)

469 Conclusion

470 This work presents a magnetophoretic system to enhance the heat exchange in a microfluidic
471 platform with an embedded heater. Experiments with CrO_2 and Fe_2O_3 nanoparticles indicate that
472 magnetically formed nanofins can be established along the walls of the microchannel upon the
473 application of a magnetic field. The nanofins are quite stable and their configurations do not
474 significantly change by increasing the flow rate from 10 to 120 $\mu\text{l min}^{-1}$. CrO_2 nanoparticles have
475 a higher thermal conductivity and are rod-shaped, and therefore the nanofins made of CrO_2
476 perform better than their Fe_2O_3 counterparts, which are comprehensively discussed in the paper.
477 The spherical Fe_2O_3 nanoparticles' cooling effect was minimal due to their low thermal
478 conductivity and high phonon scattering, which in turn reduce their efficiency. Compared to the
479 case of DI water as the coolant fluid, the formation of CrO_2 nanofins can increase the maximum
480 temperature of the hot side wall and the average temperature of the outlet by 2.5 and 2.5 K,
481 respectively, at a flow rate of 40 $\mu\text{l min}^{-1}$, and by 4 and 0.9 K, respectively, at a flow rate of
482 120 $\mu\text{l min}^{-1}$. This means that the heat conduction across the nanofins leads to better propagation
483 of heat across the platform, reducing the temperature at hot spots. The outcomes of this paper
484 show the importance of the selection type and morphology of nanoparticles in forming nanofins
485 for cooling applications.

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This work presents the thermal performance of a microfluidic system in the presence of magnetically formed nanofins (CrO_2 and Fe_2O_3).

