Soft Matter



 $\overline{\phantom{a}}$ 



# **Quantitative mechanical analysis of indentations on layered, soft elastic materials**



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 elastic modulus from indentation experiments are unable to quantitatively account for mechanical heterogeneity commonly found in biological samples. In this work, we numerically calculated force-indentation curves onto two-layered elastic materials using an analytic model. We found that the effect of the underlying substrate can be 22 quantitatively predicted by the mismatch in elastic moduli and the homogeneous-case 23 contact radius relative to the layer height for all tested probe geometries. The effect is analogous to one-dimensional Hookean springs in series and was phenomenologically modeled to obtain an approximate closed-form equation for the indentation force onto a two-layered elastic material which is accurate for up to two orders of magnitude mismatch in Young's modulus when the contact radius is less than the layer height. We performed finite element analysis simulations to verify the model and AFM  microindentation experiments and macroindentation experiments to demonstrate its ability to deconvolute the Young's modulus of each layer. The model can be broadly

used to quantify and serve as a guideline for designing and interpreting indentation

experiments into mechanically heterogeneous samples.

# **Introduction**

 Indentation-based elasticity measurements are commonly used to investigate mechanical properties of materials across many length, time, and rigidity scales.<sup>1</sup> A frequently used tool for active micro- and nanoindentation is the atomic force microscope (AFM), allowing for the characterization of materials with various probe 39 geometries with multiple magnitudes of dimensions  $(10^{-9}-10^{-5})$  m and forces  $(10^{-12}-10^{-5})$  10<sup>-6</sup> N). AFM indentation has been applied at the nanometer scale to quantify 41 mechanical properties of microtubules,<sup>2</sup> viruses,<sup>3,4</sup> polymer films,<sup>5</sup> hydrogels,<sup>6-9</sup> and is 42 being increasingly used in the fields of cell mechanics<sup>10-18</sup> and tissue mechanics.<sup>19, 20</sup> The basic principle is to indent an object with a probe of known geometry and dimensions and measure the force of indentation as a function of distance into the sample, producing a force-indentation (*F-δ*) curve. Nominally, these *F-δ* curves may be 46 fit to an elastic contact model, such as Hertz<sup>21</sup> or Sneddon<sup>22</sup>, to determine the Young's (or elastic) modulus *E* of the sample when the Poisson's ratio *ν* is known (for a detailed review, see Ref. <sup>23</sup>). *ν* is typically between 0 and 0.5 and assumed to be 0.5 for rubber- like materials. Assumptions of these models include that the sample is an infinite half- space with homogeneous, isotropic elasticity with no viscosity, there is frictionless, non-adhesive contact between the surface and the probe, and the strains are small; most samples encountered in cell and tissue mechanics do not meet these assumptions.

 One common problem is the presence of some mechanical heterogeneity in the 55 sample, for example a thin cell<sup>24</sup> or the actin cortex of a cell,<sup>25</sup> which may be modeled as a thin elastic layer that is supported by a substrate with different mechanical properties. Elastic contact models have been extended to account for the effects of an 58 infinitely rigid substrate for axially symmetric probe geometries.<sup>26-28</sup> Other models have been developed to account for a thin layer that has a small (less than one 60 magnitude) elastic mismatch to its substrate, $29-31$  define an equivalent elastic modulus

61 for a multi-layered system using a numeric approach, $32, 33$  use a mixed finite element analysis approach,<sup>34</sup> or for cells specifically to treat the top layer as an entropic 63 polymer brush and the cell body an elastic sample<sup>35</sup> or add additional contribution 64 arising from surface tension.<sup>36</sup> However, the field lacks a simple formulation that may be easily applied to any elastic mismatch between two elastic layers for all probe geometries. In this article we present a method for the mechanical quantification of elastic, layered materials, which has general application to many probe geometries and sample types. Further, it allows to estimate the influence of underlying materials to force-indentation curves and therewith the resulting Young's moduli. Finite element simulations are performed to validate the approach, and AFM microindentation and macroindentation experiments are performed to assess the applicability of the model.





 **Figure 1:** Overview of the bonded two-layer axisymmetric indentation model. (A) Illustration of the model with the relevant physical parameters. (B) Force-indentation curves calculated from the numeric model for three cases: the homogeneous case (green), a stiffer substrate (red), and a softer substrate (blue). (C) Contact radius between the indenter and surface of the indentation as a function of indentation depth. (D) Surface displacement profile calculated from Eq. 10. Parameters used to calculate are: parabolic indenter geometry (Hertz model), R=5 µm, 80 h=20  $\mu$ m, E<sub>1</sub>=100 kPa, E<sub>2</sub>=100 kPa, 10kPa, or 1MPa for green, blue, and red, respectively,  $v_1=v_2=0.5$ , and in (D) δ=1.0 µm.



 Sneddon demonstrated that the axisymmetric elastic indentation problem in cylindrical coordinates can be reduced using Hankel Transforms into a dual integral problem to solve for the stress-strain relations along the deformed surface of an infinite half-space.<sup>22</sup> From this, *F-δ* relations can be derived for any arbitrary indenter geometry defined by a function *f*. Dhaliwal and Rau extended the work of Sneddon to include an elastic layer bonded to an infinite elastic half-space (Fig. 1A) by imposing additional boundary conditions between the two elastic layers.<sup>37</sup> The *F-δ* relations are 90 determined by solving a Fredholm Integral Equation of the Second Kind: $37,38$ 

92 
$$
\phi(t) + \frac{a}{h\pi} \int_0^1 K(x,t) \phi(x) dx = -\frac{E_1 a}{2(1 - v_1^2)} [\delta - \beta(t)] \quad (1)
$$

94 
$$
F = -4 \int_{0}^{1} \phi(t) dt
$$
 (2)  
\n95  $\phi(1) = 0$  (3)  
\n96  $\phi(1) = 0$  (3)  
\n97   
\n98 where *a* is the contact radius between the probe and the sample,  $\delta$  is the probe  
\n99 indentation depth, *h* is the distance from the top of the layer to the interface with the  
\n100 substrate (height of the layer), *E*<sub>1</sub> is the Young's modulus and *v*<sub>1</sub> is the Poisson's ratio  
\n101 of the layer, *F* is the indentation force, and  $\beta$  is a function of *f* (Supplementary Table 1):  
\n102  $\beta(t) = t \int_{0}^{t} \frac{f'(ra)}{\sqrt{t^2 - r^2}} dr$  (4)  
\n104 where 0 < *r* < 1. Here  $\phi$  is an intermediate function to reduce the system of dual integral

106 equations to a single equation and is related to the stress profile of the indentation. 107 The kernel *K* is smooth and defined by

108

109 
$$
K(x,t) = 2\int_0^\infty H(2u)\cos\left(\frac{a}{h}tu\right)\cos\left(\frac{a}{h}xu\right)du \quad (5)
$$

- 110
- 111 with
- 112

113 
$$
H(u) = -\frac{d + g(1 + u)^2 + 2dge^{-u}}{e^u + d + g(1 + u^2) + dge^{-u}} \tag{6}
$$

114

115 
$$
d = \frac{(3 - 4\nu_1) - \mu(3 - 4\nu_2)}{1 + \mu(3 - 4\nu_2)}
$$
 (7)

116

117 
$$
g = \frac{1 - \mu}{\mu + 3 - 4\nu_1} \quad (8)
$$

118

119 
$$
\mu = \frac{E_1(1 + \nu_2)}{E_2(1 + \nu_1)} \quad (9)
$$

121 where *E<sup>2</sup>* and *ν<sup>2</sup>* are the Young's modulus and Poisson's ratio of the substrate. When *E<sup>1</sup>* 122 =  $E_2$  or  $h \to \infty$ , Eq. (1) is reduced to the homogeneous case considered by Sneddon.<sup>22</sup> 123 Eq. (1) is numerically solved<sup>39</sup> simultaneously with the boundary condition Eq. (3) to 124 determine *a* and *ϕ*, then *F* is calculated using Eq. (2). In the case of an indentation onto 125 a two-layer sample with a stiffer substrate  $(E_2>E_1)$ , *F* will be larger compared an 126 indentation onto a sample with homogeneous  $E_1$ , and similarly for a sample with a 127 softer substrate  $(E_2 \leq E_1)$ , *F* will be smaller compared to homogenous  $E_1$  (Fig. 1B). The 128 relationship between *a* and  $\delta$  varies in the same way as *F*; for a stiffer substrate, *a* will 129 be larger than the homogeneous case, and *a* will be smaller for a softer substrate (Fig. 130 1C). The substrate effect is strongest for *ν*=0.5 and weakens as *ν* decreases (Fig. S1). 131 This formalism does not model an asymptotic relationship in *F* when *δ*>*h*, especially 132 in the case of  $E_2>E_1$ , thus we ensure that  $\delta$ <*h* henceforth. Dhaliwal and Rau present a 133 series approximation<sup>37</sup> for  $\phi$  and we observed that this will provide a similar result as 134 Nyström interpolation<sup>39</sup> for  $E_2 \geq E_1$  but converges slowly when  $E_1 \geq E_2$  (Fig. S2).

135 Additionally, the surface deformation profile  $u_z$  of the layer as a function of 136 lateral distance from the probe is $37$ 

137

$$
u_z^1\left(r; \frac{r}{a} > 1\right)
$$

138

$$
= \frac{2}{\pi} \int_{0}^{1} \frac{\delta - \beta(x)}{\sqrt{(r/a)^2 - x^2}} dx - \frac{4(1 - v_1^2)}{\pi^2 E_1 h} \int_{1}^{r/a} \frac{dy}{\sqrt{(r/a)^2 - y^2}} \int_{0}^{1} K(y, t) \phi(t) dt
$$
\n(10)

139

140 (for *r/a* < 1, the surface strain is the difference of the probe shape and indentation). 141 The surface deformation profile depends on the elastic mismatch; the deformation at 142 higher lateral distances from the probe will increase as *E<sup>2</sup>* decreases (Fig. 1D).

 Solving the above equations will calculate *F* and *a* as a function of the other physical parameters (*f*, *δ*, *h*, *E1*, *E2*, *ν1*, *ν2*) thus providing a template to fit experimental *F*-*δ* data. The formalism has recently been used to deconvolute the Young's modulus 146 of the layer  $E_1$  using Eq. 1-9 by pre-computing a table of correction factors to the Young's modulus for an AFM dataset in which *f* is known, *δ* is constant, *F* is measured,  $v_1 = v_2 = 0.5$  is assumed, and *h* and  $E_2$  are independently measured.<sup>16</sup> We denote  $E_0$  as the apparent Young's modulus which is obtained by fitting the *F-δ* to a standard contact model (e.g. the Hertz model) and pre-compute a table of values of dimensionless

151 corrections to the apparent Young's modulus  $E_1/E_0$  as a function of the first layer height *h* and relative elastic mismatch *E1/E2*. Once the table is generated, it may be iteratively interpolated to find the unknown Young's modulus *E<sup>1</sup>* of the layer. This method directly solves Eq. (1-9) to deconvolute one of the Young's moduli, however it lacks generality and is computationally expensive as a separate table must be generated for each experimental condition and curve fitting method (for example constraints on the contact point). Thus, we seek to use numeric results of Eq. (1-9) to provide a more intuitive framework for understanding the substrate effect in the indentation problem.

## **Materials and methods**

## **Finite element analysis**

 Finite element analysis was performed using ANSYS Workbench 14.0. The models assume axial symmetry around the center of the indenting probe. The indenting probe was modeled 165 as a sphere of radius 5 µm with much higher rigidity ( $\sim$ GPa) than the samples being indented (~kPa-MPa). The sample was modeled as two bonded elastic (isotropic) layers each with *ν<sup>1</sup>* 167 =  $v_2$  = 0.49 ( $v$  = 0.49 was chosen as opposed to 0.5 for numeric stability). The layer had variable height (4 µm or 20 µm) and fixed Young's modulus (100 kPa) and the substrate had a fixed height of 200 µm and variable Young's modulus (100 Pa-100 MPa) and both layers 170 had a radius of 150 µm. The material was fixed at the bottom of the second layer and had a 171 triangular mesh size of 100 nm which and tapers to larger values at a distance of 25  $\mu$ m from the probe. The probe used a triangular mesh size of 50 nm and the contact between the tip and sample was frictionless. The probe was then moved into the sample in 2 nm increments 174 and the force response was calculated at the interface between the probe and sample, thus producing a simulated *F-δ* curve which are copied to MATLAB for analysis.

# **PDMS preparation and atomic force microscopy**

 Polydimethylsiloxane (PDMS, Sylgard 184, Dow Corning) was mixed with base:crosslinker ratios of 25:1 and 40:1 and degassed. Thin PDMS layers were spin- coated at 4000 rpm for 2 min onto silanized (Methyltrichlorosilane, Sigma Aldrich) glass coverslips (Gold Seal 48mm×60mm, Electron Microscopy Sciences) and thick (~3 mm) PDMS substrates were poured onto glass-bottom petri dishes and cured at

 65°C overnight. The layer thicknesses were determined by the interference pattern of the back reflected light at the gel interfaces using a confocal microscope (Microtime 200, PicoQuant) at 6 locations on the gel used to form the layer; thickness values 186 represent the mean  $\pm$  standard deviation. Both layers were then cleaned in oxygen plasma (PDC-001, Harrick Plasma) for 2 min, pressed together to bond, and then the silanized coverslip was removed to produce a layered PDMS substrate. AFM measurements were performed using an MFP-3D-BIO (Asylum Research) with a 10 μm diameter glass bead glued to a tipless cantilever (ACT-TL, AppNano) with stiffness 57 N/m as determined from the thermal tuning method. Experiments were performed at room temperature and in 2% bovine serum albumin in phosphate buffered saline 193 to reduce tip-sample adhesion. The probe velocity was  $2 \mu m/s$  and indentation depths up to 1.5 μm were analyzed. Data is collected from multiple indentations at different locations on a single two-layered sample as well the homogeneous stiffness samples prepared at the same time (gels within a single figure panel were prepared at the same 197 time, gels in different figure panels were not); stiffness values represent the mean  $\pm$ standard deviation from at least 48 indentations per sample.

### **PDMS preparation and macroindentation**

 PDMS was mixed with base:crosslinker ratios of 10:1 and 25:1, degassed, and cured at 65°C overnight. Bulk indentation measurements were performed with an Anton-Paar MCR302. A steel bead with a diameter of 9.53 mm was glued to a disposable measuring plate (D-PP25/AL/S07) using epoxy. The rheometer was programmed to adjust the 205 gap distance relative to the bottom of the gel (equivalent to  $\delta$  with an offset) and the resulting normal force is measured resulting in a *F-δ* curve. Measurements were performed in air. The height of the PDMS gel was determined from the contact point of the rheometer. *F-δ* curve and indentation depths up to 1.5 mm were analyzed. Data is collected from a single indentation on a single layered sample as well the homogeneous stiffness samples prepared at the same time.

### **Data analysis**

*F-δ* curves are analyzed using home-built routines in MATLAB (R2015b, MathWorks).

For all data using a spherical indenter, data is fit using linear least squares regression

- 215 on the Hertz model with a fully constrained contact point (the fits contain  $F = 0$ ,  $\delta = 0$ ).
- 216 Contact points were manually chosen in a point-and-click scheme. Eq. (1) is solved
- 217 using the MATLAB program Fie.<sup>39</sup> Built-in MATLAB functions trapz and quadv are used
- 218 for numeric integration and lsqcurvefit for curve fitting. Wolfram Mathematica 8.0 is
- 219 used to solve Eq. (4). The Poisson's ratio was assumed to be 0.5.
- 220
- 221



**223 Figure 2:** Results of the two-layer model for  $F/F_0$  for a parabolic (Hertz model) probe. (A)  $F/F_0$  as a function of 224 E<sub>1</sub>/E<sub>2</sub> for various values of a<sub>0</sub>/h (magenta a<sub>0</sub>/h=0.01, green 0.10, blue 0.25, red 0.50, and black 1.00). (B) The same 225 data as (A) except showing  $F/F_0$  as a function of  $a_0/h$  for various values of  $E_1/E_2$  (cyan  $E_1/E_2=1$ , blue 10, red 100, black 1000, green 0.1, and magenta 0.001). Open markers indicate the solution to Eq. (1-9) while solid lines 227 indicate the solution to Eq. (16).

# **Results**

### **The deconvolution method**

 In elastic contact mechanics, *F* is linearly proportional to *E* and has some non-linear 232 dependence on  $\delta$ . We denote  $F_{\theta}$  and  $a_{\theta}$  as the force and contact radius between the probe and sample for the homogeneous case of *E1* (see also Supplementary Table 1), so for the Hertz model<sup>21</sup>

236 
$$
F_0 = \frac{E_1}{(1 - {v_1}^2)^3} \sqrt{R\delta^3} \quad (11)
$$

$$
a_0 = \sqrt{R\delta} \qquad (12)
$$

 where *R* is the apex radius of the probe. In the two-layer model, *E* is replaced by *E1*, and *h* and *E2* are introduced as additional multiplicative term describe the contribution of the substrate 242 and perturbs *F* (and *a*) from the case of homogeneous  $E_1$ . In Eq. (1-9), *h* and  $E_2$  only appear in terms relative to the contact radius *a* and *E1*, respectively. Therefore, when the Poisson's 244 ratios of the layer and substrate are known (here we assume  $v_1 = v_2 = 0.5$  for incompressible

245 materials), the indentation into the two-layer material can be described as an indentation 246 into the layer treated as a homogeneous material  $E_1$  with an extra corrective term from 247 substrate. The perturbative term should have dependence on the relative height to the probe 248 size and contact radius  $a_0/h$  and the elastic mismatch  $E_1/E_2$ , both of which are dimensionless. 249 We calculated deviations from the homogeneous case  $F/F_0$  for a values of  $E_1/E_2$  in the 250 range of  $10^{-3}$  to  $10^{3}$  and  $a_{0}/h$ <1 for a parabolic (Hertz model) indenter (Fig. 2A, B). When 251 *E*<sub>1</sub> $\lt E_2$ ,  $F/F_0$ >1 and the substrate effect saturates for increasing  $E_2$ , indicating an upper limit of 252 *E<sub>2</sub>* altering the *F*- $\delta$  response. When  $E_1 > E_2$ ,  $F/F_0 < 1$  and the effect diverges for decreasing  $E_2$  and

253 the *F-δ* response becomes dominated by the rigidity of the substrate. These effects are 254 amplified as  $a_0/h$  increases. We also calculated  $F/F_0$  for various  $E_1/E_2$  and  $a_0/h$  for other 255 common axisymmetric indenter geometries: a cone (Sneddon model<sup>22</sup>), hyperbola,<sup>38</sup> and 256 cone with a spherical cap (sphero-cone,<sup>16</sup> Supplementary Table 1), and these are similar for 257 each geometry (Fig. S3).

258 The shape of *F/F0* for the 3D axisymmetric indentation is similar to the 1D problem 259 of compressing springs with stiffnesses *k1* with *k2* in series (Supplementary Text); in the 260 regime of  $k_2$ >> $k_1$  the force required saturates to compress only  $k_1$  without the second spring, 261 and in the regime of *k1*>>*k2* a power law emerges (Eq. S3). We phenomenologically extend the 262 1D solution to the full 3D solution using Eq. (1-9) and assuming  $v_1 = v_2 = 0.5$  to take the form of 263

$$
B\left(\frac{a_0}{h}\right) + 1
$$
\n
$$
F \approx F_0
$$
\n
$$
B\left(\frac{a_0}{h}\right)\left(\frac{E_1}{E_2}\right)^{A\left(\frac{a_0}{h}\right)} + 1
$$
\n(13)

265

266 where 0≤*A*≤1 is the power law behavior in the elastic mismatch and *B* is the saturation 267 point at  $E_1 \ll E_2$ , both of which depend on the 3D geometry  $a_0/h$ . In the 1D case of two 268 springs in series,  $A = B = 1$ . We fit the data in Fig. 2 to Eq. (13) as a function of  $E_1/E_2$  and 269 subsequently fit *A* and *B* as functions of *a0/h* to obtain (Supplementary Note 1, Fig. S4) 270

271 
$$
A\left(\frac{a_0}{h}\right) \approx \min\left(1, 0.72 - 0.34\left(\frac{a_0}{h}\right) + 0.51\left(\frac{a_0}{h}\right)^2\right) \tag{14}
$$

273 
$$
B\left(\frac{a_0}{h}\right) \approx 0.85\left(\frac{a_0}{h}\right) + 3.36\left(\frac{a_0}{h}\right)^2 \quad (15)
$$

274

275 These satisfy  $F=F_0$  when  $E_1=E_2$  or  $a_0/h=0$  and  $F\approx F_0E_2/E_1$  when  $a_0 >> h$ . The value and 276 interpretation of *B* is similar to other layered indentation models with a rigid substrate 277 ( $E_2$ =∞).<sup>26</sup> Eq. (14-15) are valid only for  $a_0/h$  < 1 (Fig. S4C) and errors arise for large 278  $a_0/h$  and  $E_1 >> E_2$ . Eq. (14-15) were approximated for a parabolic (Hertz model) 279 indenter, however the numeric coefficients will be similar for other indenter shapes 280 up to  $a_0/h \sim 0.5$ . Thus, we can write a final equation to approximately fit experimental 281 *F-δ* data using a spherical or parabolic indenter on a two-layered sample as

282 
$$
F \approx \frac{16E_1\sqrt{R\delta^3}}{9} \left( \frac{0.85\left(\frac{a_0}{h}\right) + 3.36\left(\frac{a_0}{h}\right)^2 + 1}{\left(0.85\left(\frac{a_0}{h}\right) + 3.36\left(\frac{a_0}{h}\right)^2\right)\left(\frac{E_1}{E_2}\right)^{0.72 - 0.34\left(\frac{a_0}{h}\right) + 0.51\left(\frac{a_0}{h}\right)^2}{+ 1} + 1} \right)
$$
(16)

283

284 where,  $v_1 = v_2 = 0.5$  is assumed,  $\delta < h$ , and  $a_0/h < 1$ . Eq. (16) as written provides a similar 285 result as numerically solving Eq. (1-9) to within 8% for  $a_0/h < 0.8$  and  $E_2 > 10E_1$ , or 286 a<sub>0</sub>h<0.5 and E<sub>2</sub>>100E<sub>1</sub> (Fig. 2). *F* depends on all parameters ( $E_1$ ,  $E_2$ , and  $h$ ) and the 287 effects of changing one variable may be similar to changing another; thus, fitting for 288 multiple unknown parameters in Eq. (16) may result in overfitting of the data and 289 errors in determining any of the input parameters will propagate into further errors 290 in fitting for the unknown parameters. Henceforth in this article, fitting for either  $E_1$  or 291 *E<sup>2</sup>* is performed when the other modulus is known and *h* is always known (single 292 unknown parameter fitting). 293



294

295 **Figure 3:** Finite element analysis validation of the two-layer indentation model. (A) F-δ curves generated from 296 the numeric model for the homogeneous case (green), a softer substrate (blue,  $E_2=E_1/10$ ), and a stiffer substrate 297 (red,  $E_2=10E_1$ ), and the corresponding F- $\delta$  curves calculated from Eq. (1-9) (solid lines). Parameters used are 298 E<sub>1</sub>=100 kPa, h=4 µm, R=5 µm,  $\delta$ =1.0 µm, and  $v_1=v_2=0.49$ . (B) Fits for the Young's modulus of a layered material 299 with a stiff substrate  $(E_2=10E_1)$ , including the Hertz model fit (red), the fit for the layer  $E_1$  (magenta), the fit for 300 the substrate E<sub>2</sub> (cyan), and the fit for the layer E<sub>1</sub> of a homogeneous material (green). (C) Same as (B) for a layered  $301$  material with a soft substrate (E<sub>2</sub>=E<sub>1</sub>/10). (D) Deconvolutions of finite element analysis data using precomputed  $302$  correction tables. Plus signs show fits using the standard Hertz model E<sub>0</sub>, circles are two-layer fits for the layer E<sub>1</sub>  $303$  which is kept constant at 100 kPa, squares are fit for the substrate E<sub>2</sub> which is variable (simulation moduli shown 304 in black), blue denotes a thicker layer (h=20  $\mu$ m), red denotes a thinner layer (h=4  $\mu$ m). (E) Deconvolutions of 305 finite element analysis data by performing a least squares fit on Eq. (14), using the same legend as (C) with  $\delta$  up  $306$  to  $a_0/h=0.10$ ,  $a_0/h=0.25$ , and  $a_0/h=0.50$  in blue, green, and red, respectively.

# 308 **Validation of the method with simulated F-δ curves**

 To test the validity of the model, we performed finite element analysis simulations of an indentation into a layered elastic material using a spherical probe. We simulated *F-δ* data for two different layer heights (20 μm and 4 μm, corresponding to *a0/h*=0.11 312 and  $a_0/h=0.56$ , respectively, for a spherical probe with  $R=5 \mu m$  and  $\delta=1 \mu m$ ) with varying *E<sup>2</sup>* and constant *E<sup>1</sup>* (Fig. 3C). We observed good agreement in the *F-δ* curves generated by finite element analysis and the two-layer model Eq. (1-9) for the same 315 geometric and material parameters (Fig. 3A) and in the surface displacement profile 316 of the layer with Eq. (10) (Fig. S5). When we fit the *F-δ* curves on a layered material to 317 the Hertz model to determine  $E_0$ , we observe a dependence of the fitted (apparent) 318 Young's modulus value with indentation depth (Fig. 3B, C). Fitting *F-δ* data using Eq.  $319$  (16) for  $E_1$ , removes the depth-dependence of the Young's modulus and we can 320 recover the value of the layer in the simulation. Furthermore, fitting the *F-δ* data using 321 Eq. 16 for  $E_2$  provides mixed results; when  $E_1 \ge E_2$  we are able to recover the value of  $E_2$ 322 in the simulation (Fig. 3C), however when  $E_1 > E_2$  we are only able to recover the value 323 at high indentation depths (Fig. 3B). This is due to the fact that *F/F<sup>0</sup>* will plateau at a 324 certain value of  $a_0/h$  even if  $E_2$  is further increased (Fig. 2A).

325 We next fit for the Young's moduli of the two layers from finite element models 326 for different combinations of  $E_1/E_2$  and  $a_0/h$  (Fig. 3D, E). In the case of an indentation  $327$  into a thick ( $h=20 \mu m$ ) layer, the standard Hertz model is able to accurately estimate 328 the layer Young's modulus in the case of  $E_2 \ge E_1$ , however for  $E_1 \ge E_2$ , large errors arise 329 which scale with decreasing  $E_2$ . When the layer height is thin (h=4  $\mu$ m), errors arise in 330 estimating the Young's modulus using the Hertz model when  $E_2 > E_1$  and are more 331 pronounced for  $E_1 \ge E_2$  (Fig. S6A). When we directly used Eq. (1-9) as a template to fit 332 the simulated finite element data (Fig. 3D), we obtained *E<sup>1</sup>* for nearly all *h* and *E2*, and 333 *E<sub>2</sub>* for  $E_2 \le E_1$  within 20% accuracy, however  $E_2$  could not be estimated when  $E_1 \le E_2$  due 334 to the saturating effect on *F/F<sup>0</sup>* with increasing *E2*.

335 The deconvolution results for the approximate method Eq. (16) (Fig. 3E, S6B) 336 provides similar results however with less accuracy for large  $a_0/h$  and small  $E_2$ ;  $E_1$  has 337 errors within  $\sim$  20% when the elastic mismatch is within two orders of magnitude and 338 *a*<sub>0</sub>/h<0.25, and  $E_2$  cannot be determined when  $E_2 > E_1$ . Errors in determining  $E_1$  in the 339 regime of  $E_1 \ge E_2$  become larger than the exact method Eq. (1-9) as  $E_2$  decreases and 340 *a0/h* increases.



 **Figure 4:** Experimental indentation-based measurements of layered PDMS gels. (A) Schematic of AFM indentation of a layered PDMS gel.(B) *F-δ* curves of a soft layer and rigid substrate (green), a homogenous 346 soft gel (blue), and a homogenous stiff gel (red). Fits are shown using Hertz model ( $E_0$ , in black) for the homogenous gels, and the two-layer model(*E1*, in black) with known substrate and the substrate modulus (*E2*) with known layer height. (C) *F-δ* curves for a stiff layer and soft support (magenta) and similar analysis as in B. The two-layer model was used to fit for the layer modulus (*E1*) when the substrate modulus and layer height are known, and is also used to fit the substrate modulus (*E2*) when the layer modulus and layer height are known. (D) Schematic of macroindentation on a layered PDMS gel. (E) *F-δ* curves showing rheometer data for single-layered gels and double-layered gels with the respective Young's modulus values from fitting.

### **Demonstration of the deconvolution method with experimental F-δ curves**

 In order to test the efficacy of the model in analyzing experimental *F-δ* data, we fabricated layered PDMS samples with stiffness mismatch (base:crosslinker ratios of 25:1 and 40:1 for stiff on soft, respectively) and performed AFM-based microindentation (Fig. 4A) with a spherical probe (Fig. S7). Here we measure the Young's moduli of both layers independently and the layer thickness using the 361 interference patterns of the back reflected light at the interfaces  $(h \sim 17 \mu m)$  and  $δ=1.5$  $\mu$ m, thus  $a_0/h \sim 0.16$ ). Once the modulus of each homogenous gel had been independently determined, we tested if we could use Eq. (16) to deconvolute each constituent modulus in a layered sample when one of *E1*, *E2*, or *h* is fitted for and the other two are treated as known input parameters. In the case of a soft layer with stiff

366 substrate  $(E_1 \le E_2)$ , we observe that the force is higher than the homogenous soft gel and less than the homogenous stiff gel (Fig. 4B), thus fitting with the Hertz model for 368 *E*<sub>0</sub> (58  $\pm$  6 kPa) provides a result different from both (258  $\pm$  8 kPa and 49  $\pm$  4 kPa for 25:1 and 40:1, respectively). When the force-indentation curve on the layered gel is fit 370 with Eq. (16) with known substrate  $E_2$  (the value of  $E_0$  of the homogeneous stiff gel) 371 and *h*, the fitted value  $E_1$  (51  $\pm$  5 kPa) of the layer is in good agreement with the homogenous stiffness *E<sup>0</sup>* of the soft gel. We do not fit for *E<sup>2</sup>* as we observed that fitting 373 for  $E_2$  in the case of a stiff substrate with a relatively low  $a_0/h$  results in high errors in determining *E2*.

 The effect is similar for the case of a stiff layer and soft substrate (Fig. 4C); the 376 *E*<sub>0</sub> fits for these samples differ from both homogenous gels (196  $\pm$  15 kPa, 226  $\pm$  5 kPa, 377 and  $81 \pm 3$  kPa for layered, 25:1, and 40:1, respectively). However, when  $E_2$  ( $E_0$  of the homogenous soft gel) and *h* are used as input parameters in Eq. (16), the fitted value 379 *E*<sub>1</sub> (230  $\pm$  22 kPa) is similar to  $E_0$  of the stiff gel. When  $E_1$  ( $E_0$  of the homogenous stiff 380 gel) and *h* are used as input parameters in Eq. (16), the fitted  $E_2$  (93 ± 29 kPa) is similar 381 to  $E_\theta$  of the soft gel.

 As the model is applicable to any length scale, we additionally performed macroindentation experiments on layered PDMS gels using a steel bead (*R* = 4.8 mm) glued to the measuring plate of a rheometer (Fig. 4D). We collected *F-δ* data on two- layered and single-layered PDMS gels with crosslinking ratio of 10:1 and 25:1 and fit the data using Eq. (14) (Fig. 4E). Indentations were first performed on homogenous PDMS gels and we obtained Young's moduli *E<sup>1</sup>* of 1.39 and 0.26 MPa for 10:1 and 25:1 crosslinking ratios, respectively. Thin (*h* ~ 9 mm and *δ* = 1.5 mm, thus *a0/h* ~ 0.3) slices of additional PDMS gel were attached on top of the thick gels and additional *F-δ* data (Fig. 4E) was collected. As qualitatively expected, the force of indentation on a thin 10:1 gel with a 25:1 substrate is lower than the force of indentation on a homogeneous 10:1 gel, and the force of indentation on a thin 25:1 gel with a 10:1 substrate is higher than the force of indentation on a homogeneous 25:1 gel. Fits for *E<sup>0</sup>* using the Hertz model gave results that are significantly different from the homogenous gel stiffnesses (0.72 MPa for 10:1 on top of 25:1; 0.38 MPa for 25:1 on top of 10:1). When the *F-δ* data 396 is analyzed using Eq. (16), the fit for  $E_1$  of the layered gels gives values comparable to those measured on the bulk gels (1.10 and 0.29 MPa for 10:1 and 25:1 thin layers, 398 respectively). In the case of the thin 10:1 gel on the 25:1 gel substrate, because  $E_2 \le E_1$ 

- itis possible to fitfor *E2*, and we obtained 0.18 MPa for the thick 25:1 substrate beneath
- the 10:1 thin layer. Taken together, these experiments serve as a proof-of-principle for
- using the model presented here in quantitatively analyzing *F-δ* curves on elastic
- layered samples.
- 





406 **Figure 5:** Comparison of *F/F<sup>0</sup>* with other layered models. (A) The case of a parabolic indenter and a rigid 407 substrate; blue shows the numeric solution of Eq. (1-9), red shows Eq. (16), yellow shows the model by 408 Dimitriadis et al.,<sup>27</sup> and magenta shows the model by Garcia and Garcia.<sup>26</sup> (B) The case of a conical 409 indenter and a rigid substrate; blue shows the numeric solution Eq. (1-9), red shows Eq. (16), yellow 410 shows the model by Gavara and Chadwick,<sup>28</sup> and magenta shows the model by Garcia and Garcia.<sup>26</sup> (C) 411 The case of a parabolic indenter and a two-layer substrate as a function of *E1/E2*; open markers indicate 412 the model by Hsueh and Miranda<sup>30</sup> and solid lines indicate Eq. (1-9) with a*0/h* values of 0.01 (red), 0.10 413 (green), 0.25 (blue), 0.50 (magenta), and 1.00 (black).

# 415 **Discussion**

416 Indentations into bonded two-layered elastic systems are treated as indentation into 417 a homogeneous layer with an additional perturbative term that depends on *E1/E<sup>2</sup>* and 418 *a0/h* and summarized as Eq. (16). Qualitatively, the force of indentation scales in a 419 similar manner to springs in series. For a stiffer substrate  $(E_2>E_1)$ , the effect of the 420 substrate will increase with  $a_0/h$  but will saturate for higher values of  $E_2$ , and the 421 saturation point will depend on the value of *a0/h*. In this scenario, the dominant factor 422 in the perturbative term is  $a_0/h$ , which directly relates to the indentation depth. For a 423 softer substrate  $(E_2 \leq E_1)$ , there is a power-law relationship between  $F/F_0$  and  $E_1/E_2$ . In 424 this scenario, both  $E_1/E_2$  and  $a_0/h$  strongly affect  $F/F_0$ .

 The two-layer model presented here may be used as a quantitative guide during the design and analysis of AFM indentation experiments with biological samples. If elastic heterogeneity is known or suspected (for example a mammalian cell with stiff actomyosin cortex,<sup>25</sup> a thin cell or polymer gel, or a cell seeded on soft 429 extracellular matrix<sup>16, 40</sup>),  $E_1/E_2$  and  $a_0/h$  may be roughly estimated and Eq. (16) will predict two-layer effects on *F*. Depending on the scientific question being addressed, *a*<sub>0</sub> can be tuned by changing  $\delta$  or the indenter geometry. If the goal is to quantify the 432 rigidity of the layer  $(E_1)$ , using low  $a_0/h$  will help to ensure that the layer dominates 433 the *F-δ* response. If the goal is the measure the rigidity of the substrate  $(E_2)$ , then

434 increasing  $a_0$  will make the *F-δ* response more characteristic of  $E_2$  (Fig. 2, S4C), 435 however the model presented here is inaccurate for  $E_2 >> E_1$  and another approach 436 where  $\delta$ >h would be more suitable, such as in Kaushik et al.<sup>19</sup> As the two-layer effect depends on all of the sample parameters (*h*, *E1*, *E2*) as well as the indentation parameters (*f*, *δ*) (Fig. 2), it may be necessary to accurately and independently measure several of these parameters or perform further experiments on the same sample using different probe geometries in order to accurately determine the unknown parameter. For a practical example, it has been observed that a cell's Young's modulus measured by AFM depends on the geometry of the probe, with sharper 443 probes resulting in larger values.<sup>18, 25</sup> As the substrate effect scales with the contact radius, the two-layer model corroborates observations that sharper conical probes are more sensitive than larger bead probes to the stiffer actomyosin cortex than the cell body.<sup>25</sup>

 In Hertzian contact with a parabolic indenter there is power-law scaling *F*∝*δ 3/2* , however if a second layer is present then this power-law behavior is changed; for  $E_2 \geq E_1$  the exponent will be effectively higher and for  $E_2 \leq E_1$  it will be lower (Fig. 1B). If an experimental *F-δ* curve bends from the expected power-law behavior and the fitted elastic modulus depends on indentation depth (as in Fig. 2B, C) then mechanical heterogeneity may be present; the two-layer model provides guidelines for interpreting this data.

 Other analytical models have been independently derived in the case of indentations onto thin samples with a rigid substrate. The model presented here 456 provides similar corrective terms as Dimitriadis et al.<sup>27</sup> and Garcia and Garcia,<sup>26</sup> 457 although differ from Gavara and Chadwick<sup>28</sup> (Fig. 5A, B). Both the two-layer model here and by Garcia and Garcia<sup>26</sup> observe a direct dependence of both *F* and *a* on *a0/h* for multiple probe geometries. We also compared our two-layer model with another 460 derived by Hsueh and Miranda<sup>30</sup> and find that the corrective terms agree only for small stiffness mismatches (Fig. 5C). It should be noted that Eq. 16 was approximated to examine the substrate effects on indenting the top layer, analogous to a "bottom-effect 463 correction". When  $E_2 \ll E_1$  or  $a_0/h > 1$ , the force becomes more characteristic of the substrate (e.g. the top layer effects on indenting the substrate) and Eq. (16) is no longer accurate (Fig. S4C). Future work could be performed to improve the

 phenomenological approach in Eq. (13-16) to account for the asymptotic switch 467 between the two mismatched layers, such as in Korsunsky and Constantinescu.<sup>32</sup>

 Finally, the model also predicts that, as *E<sup>2</sup>* decreases, the shape of the deformed surface of the layer will change such that positions further away from the probe will undergo higher displacement. This also means that the 3D strain field induced by a 471 deformation when  $E_2 < E_1$  will be larger than the homogeneous case. The two-layer model presented here is only valid when the substrate (an elastic half-space) and the layer have infinite lateral dimensions. However, based on the surface displacement Eq. (10), the model predicts that the effects on *F* from a finite-sized sample will be 475 amplified when  $E_2 \leq E_1$ .

## **Conclusions**

 We describe a simple method for the quantitative mechanical analysis of two-layered elastic materials and anticipate that our results will yield to more precise quantification of heterogeneous soft matter and biological specimens. This model may be easily applied to the design, analysis, and interpretation of AFM indentation experiments and may also be used as a general description of elastic contact mechanics regarding the correlation of length scales and stiffness in the presence of external forces and deformations. Considerations for future improvements include quantitative effects of lateral sample size and viscous and non-linear effects of layered materials.

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