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Physical Chemistry Chemical Physics

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This article can be cited before page numbers have been issued, to do this please use: D. W. W. Nielsen and C. Hélix-Nielsen, *Phys. Chem. Chem. Phys.*, 2026, DOI: 10.1039/D5CP03542B.



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Cite this: DOI: 00.0000/xxxxxxxxxx

Received Date

Accepted Date

DOI: 00.0000/xxxxxxxxxx

Measuring the Gibbs Free Energy of Mixing Using the Concept of an Osmotic Engine.

Dennis Wowern Nielsen^{*a} and Claus Helix-Nielsen^b

This manuscript develops a membrane-thermodynamic framework for quantifying the Gibbs free energy of mixing in real binary solutions. Using an osmotic engine representation under pressure retarded osmosis, the method links static and dynamic membrane-based osmometry to the excess Gibbs free energy of mixing, enabling operational estimation of ΔG_E from measurable osmotic and volumetric responses. As an analytical perspective, the van't Hoff limit produces a simple colligative form that connects the Henry-reference activity coefficient at infinite dilution to the dimensionless osmotic pressure, highlighting the role of reference-state regularisation and the sensitivity to volume assumptions. Beyond this limiting case, the framework provides a natural route to multicomponent mixtures with electrolytes and non-electrolytes, predicts solvent-dependent shifts in reaction equilibrium constants, and supports routine estimation of the Flory-Huggins solvent-polymer interaction parameter.

1 Introduction

In the realm of chemical phenomena taking place in the liquid phase, assessment of Gibbs free mixing energy, ΔG_{mix} , holds a pivotal position. When examining a chemical equilibrium reaction within an inert solvent, one may define a series of steps: Firstly, the dissolving of reactants, inducing a change in Gibbs free energy, ΔG_{mix}^{react} . Subsequently, a reaction must take place, leading to the formation of reaction products, which, in turn, results in a change in Gibbs free energy, ΔG_{rxn} . Finally, the reaction products dissolve within the mixture, leading to yet another term, ΔG_{mix}^{prod} . Thus at equilibrium

$$-\Delta G_{rxn} = \Delta G_{mix}^{react} + \Delta G_{mix}^{prod} \quad (1)$$

Hence, the energies of reaction are inseparable from energies of mixing, and like this it is argued that the measurement of the chemical mixing energy is not merely supplementary to, but in many cases as essential as, the knowledge of equilibrium constants for understanding specific chemical reactions.

Consider for example a solution of protein P and ligand L dissolved in a suitable solvent, typically water, entering the chemical addition reaction $P + L \rightleftharpoons PL$. At equilibrium, the chemical affinity

constant, K_a , is related to a set of thermodynamic parameters

$$-n_{PL}RT \ln K_a = (n_P - n_{PL})\Delta\bar{G}_{mix}^P + (n_L - n_{PL})\Delta\bar{G}_{mix}^L + n_{PL}\Delta\bar{G}_{mix}^{PL} + n_{PL}(\mu_{PL}^\circ - \mu_P^\circ - \mu_L^\circ) \quad (2)$$

Here, n represents the amount of substance of reactants or products, R is the universal gas constant, T is the kelvin temperature, $\Delta\bar{G}_{mix}$ signifies the change in partial molar Gibbs free mixing energy of the reaction's products or reactants, and μ° represents the standard chemical potential of formation for the reaction's products or reactants. Since the formation potentials remain constant and independent of the inert solvent, mixing energies directly influence K_a , making it a system-dependent constant. This implies that by determining both the mixing energies and formation energies for each reactant and product, the equilibrium constant may be determined.

If the reaction is predominantly shifted towards the product side, Eq.(2) can be further simplified as follows

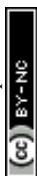
$$-RT \ln K_a = \Delta\bar{G}_{mix}^{PL} + \mu_{PL}^\circ - \mu_P^\circ - \mu_L^\circ \quad (3)$$

for $n_P = n_L$. Hence, by performing the measurements in a series of different inert solvents, it is demonstrated that, at constant temperature and pressure, $-RT \ln K_a$ is linearly dependent on $\Delta\bar{G}_{mix}^{PL}$. Consequently, if $-RT \ln K_a$ is determined from ITC measurements in these various solvents and the corresponding $\Delta\bar{G}_{mix}^{PL}$ values are measured, the typically inaccessible quantity $\mu_{PL}^\circ - \mu_P^\circ - \mu_L^\circ$ can be estimated.

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Declaration: The authors declare no conflicts of interest. Dennis Wowern Nielsen wrote the first draft, and Dennis Wowern Nielsen and Claus Helix-Nielsen contributed equally to the final version.



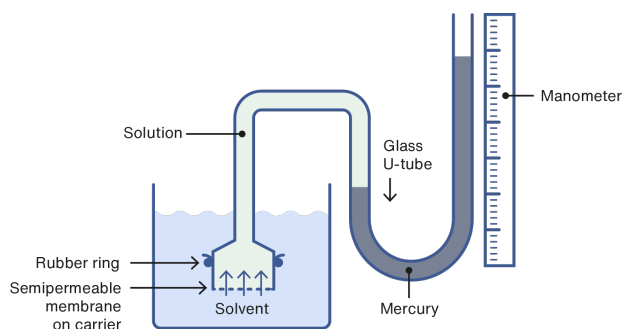


Fig. (1) Principal sketch of an arrangement performing *static* osmometry.

In summary, it is proposed that a reliable and precise measurement of ΔG_{mix} across different inert solvents could significantly advance chemical research, and since $dG = VdP$ at constant temperature, this method is recommended for further exploration in the context of osmometry.

Today's commercial osmometers operate based on three distinct principles: Freezing point depression measurement, which relies on osmotically active compounds lowering the freezing point of a solution; vapor pressure measurement, where osmotically active particles reduce the vapor pressure of a solution; and osmotic pressure measurement, which involves isolating a solution from pure solvent using a semipermeable membrane. While techniques such as freezing point depression and vapor pressure osmometry are commonly used to determine osmotic pressure differences, they are insufficient for quantifying the volume displaced at osmotic equilibrium¹. As a result, these methods cannot estimate the work performed on the surroundings, which is represented by the product of the displaced volume and the osmotic pressure of the solution prior to osmosis, arising from the spontaneous osmotic dilution process.

To overcome this limitation, conventional membrane based osmometry is recommended, as it enables the simultaneous measurement of both osmotic pressure and displaced volume. This approach allows for the determination of osmotic work exerted on the surroundings through two distinct methods: The *static* and *dynamic* equilibrium methods².

Pfeffer³ pioneered the development of membrane-based osmometers, enabling the simultaneous measurement of osmotic pressure and displaced volume using the *static* method, as illustrated in Fig. (1). This approach facilitates the establishment of osmotic equilibrium for evaluating the pressure difference. The resulting pressure, when multiplied by the volume change in the solution compartment, quantifies the work exerted on the surroundings.

However, highly concentrated, strongly non-ideal solutions can generate osmotic pressures that, at equilibrium, lead to liquid column heights on the order of hundreds of meters, rendering them impractical for standard laboratory settings. Since osmotic equilibrium is a reversible process that may take an indefinite amount of time to establish, accurately measuring osmotic pressure and volume change remains a challenge.

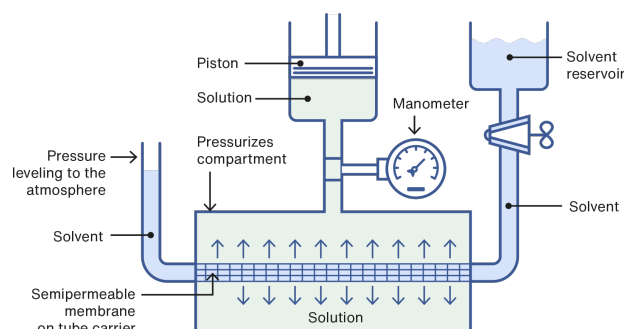


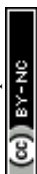
Fig. (2) Principal sketch of an arrangement performing *dynamic* osmometry.

Moreover, this technique is influenced by several factors, including membrane material, surface area, thickness, selectivity, solvent and solute permeability, and the membrane's susceptibility to complete or partial swelling in the solvent. Additionally, ambient pressure and temperature introduce uncertainties. These variables may fluctuate over time, complicating the determination of when osmotic equilibrium is achieved. For instance, the kinetics of osmotic pressure development in aqueous solutions, depending on sucrose solute concentration, vary significantly under two conditions: Constant and variable solution volumes⁴. The rate of pressure increase is notably higher under constant volume conditions than under variable volume conditions, primarily due to the significantly greater solvent flow in the latter. This effect becomes more pronounced as sucrose concentration increases, exemplifying the behavior of real sucrose solutions.

This challenge underscores the necessity of employing *dynamic* membrane osmometry, where the extended time required for solvent volume flow is reduced by artificially increasing the pressure difference. Consequently, relying solely on displaced volume and osmotic pressure to evaluate the work in osmotic dilution processes using *static* membrane osmometry may be insufficient for determining the work done on the surroundings.

In the *dynamic* equilibrium method, initially pioneered by Berkeley-Hartley⁵, a counterpressure is applied to nullify any consequential volume flux between the solvent and solution chambers, see Fig. (2). Although this technique facilitates the expeditious determination of the solution's osmotic pressure, it regrettably fails to furnish the requisite information regarding the volume expansion that an unobstructed osmotic dilution would have incurred. Consequently, the assessment of osmotic displacement work remains unattainable via the *dynamic* method.

To address the challenges associated with quantifying both the osmotic pressure and the volume displaced at osmotic equilibrium, this publication introduces the concept of an osmotic engine, as illustrated in Fig. (3). This is a theoretical thermodynamic framework that effectively merges *static* and *dynamic* membrane-osmometry techniques. The approach enables the computation of the Gibbs free energy of mixing of a real binary solution and the corresponding excess Gibbs free energy, and it also allows determination of the solute activity coefficient, among other applications. The concept engine is formulated as a cascade of cylinders that, in each stage, processes an initial volume αV_C ,



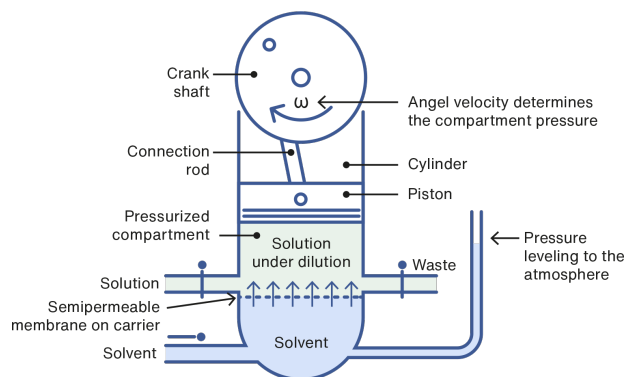


Fig.(3) Principal sketch of a concept arrangement which combines *static* and *dynamic* osmometry.

where V_C is the volume of a single cylinder and $0 < \alpha < 1$. To ensure that subsequent physico-chemical conclusions, e.g. expressions for ΔG_{mix} and the solute activity coefficient, are apparatus-independent rather than tied to a particular discrete engine concept design, the continuum limit $\alpha \rightarrow 1$ is considered. In this limit, the per-stage increment becomes vanishingly small and the cascade becomes effectively continuous, corresponding to the idealized limit of infinitely many cylinders.

2 Theory

In the following, the binary *Test* solution is treated as a practically incompressible liquid, so that volume changes during osmotic dilution are ascribed to changes in composition rather than to elastic compression. This keeps the modelling of ΔG_{mix}^{Test} focused on mixing non-ideality rather than on the ideal or non-ideal elasticity of the liquid.

A binary solution consisting of solvent *A* and solute *B* at constant temperature T and at the standard pressure P° is considered. This concentrated *Test* solution containing $n(A)$ and $n(B)$ is osmotically diluted to equilibrium by adding an additional amount $n^{Dil}(A)$ of *A*, yielding a more dilute solution containing $n(A) + n^{Dil}(A)$ and $n(B)$, hereafter denoted $Test^{Dil}$.

The *Test* solution is regarded as a high-concentration state, whereas the final state $Test^{Dil}$ represents a low-concentration state. The controlled dilution process connecting these two states is viewed as a chemical analogue of a Carnot process: The difference in mixing free energy between the concentrated and the dilute state is converted into mechanical work by an osmotic engine, while the Gibbs free energy of mixing of the solution changes.

The purpose of this section is to establish a simple operational relation between $\Delta G_{mix}(n(A), n(B)) = \Delta G_{mix}(Test) = \Delta G_{mix}^{Test}$, the Gibbs free energy of mixing in the initial *Test* solution, $\Delta G_{mix}(n(A) + n^{Dil}(A), n(B)) = \Delta G_{mix}(Test^{Dil}) = \Delta G_{mix}^{Final}$, the Gibbs free energy of mixing in the final osmotically diluted solution, and the reversible work associated with the osmotic dilution.

2.1 Energy balance in the osmotic engine

The two-state balance is written as (arguments are omitted for brevity)

$$\Delta G_{mix}^{Test} - \Delta G_{mix}^{Final} = -\Delta G_{sys}^{Dil} \quad (4)$$

For convenience, the quantity

$$\Delta G_{Dil} \equiv -\Delta G_{sys}^{Dil} \quad (5)$$

is defined as the dilution work, i.e. the maximum reversible work that can be delivered by the osmotic engine during dilution. For a spontaneous dilution process, $\Delta G_{sys}^{Dil} < 0$ is obtained and therefore $\Delta G_{Dil} > 0$ is obtained.

Throughout this work, a distinction is made between the system free-energy change upon dilution and the work extracted by the osmotic engine. The system free-energy change is defined as $\Delta G_{sys}^{Dil} \equiv G_{Final} - G_{Test}$, and the maximum reversible work delivered is defined as $\Delta G_{Dil} \equiv -\Delta G_{sys}^{Dil}$. With this convention, $\Delta G_{Dil} > 0$ is obtained for spontaneous dilution, and

$$\Delta G_{mix}^{Test}(n(A), n(B)) = \Delta G_{mix}^{Final}(n(A) + n^{Dil}(A), n(B)) + \Delta G_{Dil} \quad (6)$$

This relation constitutes a chemical analogue of the Carnot balance between two reservoirs and a machine: The decrease in the system Gibbs free energy upon dilution is converted into reversible mechanical work delivered by the osmotic engine.

The dilution from $(n(A), n(B))$ to $(n(A) + n^{Dil}(A), n(B))$ is assumed to be implemented in an osmotic engine. On the *B*-side of an ideal semipermeable membrane, permeable only to *A*, the solution is present, whereas pure *A* is present on the opposite side. On the *B*-side, the pressure is increased by an externally imposed counterpressure ΔP , such that the total pressure is given by $P^\circ + \Delta P$. The volume of the solution in the *B*-chamber is denoted V , with $V(Test)$ corresponding to the *Test* state and $V(Final)$ corresponding to the final diluted state. During an infinitesimal dilution step, the volume on the *B*-side is increased by dV_{Exp} due to osmotic influx of *A*.

According to the Kedem–Katchalsky formulation of volumetric flux through a semipermeable membrane, combined with Onsager's linear irreversible thermodynamics^{12–14} and Gibbs' maximum work theorem at constant T and P° , the reversible mechanical work delivered by the osmotic engine in such a step is given by

$$\delta W_{rev} = \Delta P \cdot dV_{Exp} \quad (7)$$

and this work is related to the differential change in the Gibbs free energy of the closed system by

$$\delta W_{rev} = -dG_{sys} \quad (8)$$

Along a quasi-reversible Kedem–Katchalsky path from $V(Test)$ to $V(Final)$, for which $J_V \rightarrow 0$ is satisfied at each intermediate state and the actual osmotic pressure, $\Delta\pi = [\Delta\pi]$, approaches the applied counterpressure ΔP , integration yields

$$\Delta G_{sys}^{Dil} = G_{Final} - G_{Test} = - \int_{V(Test)}^{V(Final)} \Delta P(V) dV \quad (9)$$

where $\Delta P(V)$ denotes the measured pressure difference as a func-



tion of the solution volume V . With the definition $\Delta G_{Dil} = -\Delta G_{sys}^{Dil}$, the dilution work is obtained as

$$\Delta G_{Dil} = \int_{V(Test)}^{V(Final)} \Delta P(V) dV \quad (10)$$

By insertion into the two-state balance for the mixing free energy, the central operational expression is obtained as

$$\Delta G_{mix}^{Test}(n(A), n(B)) = \Delta G_{mix}^{Final}(n(A) + n^{Dil}(A), n(B)) + \int_{V(Test)}^{V(Final)} \Delta P(V) dV \quad (11)$$

In this representation, the analogy to a Carnot process is rendered transparent: ΔG_{mix}^{Test} and ΔG_{mix}^{Final} play the role of two chemical levels, whereas the integral $\int \Delta P dV$ represents the work term, analogous to the area under a pressure–volume curve in a mechanical cycle. For some systems, $\Delta P(V)$, which coincides with the osmotic pressure $\Delta\pi(V)$ at equilibrium, can be represented analytically. Examples include van't Hoff ideality in the dilute limit

$$\Delta\pi^{vH}(V) = \frac{n(B) \cdot RT}{V} \quad (12)$$

or $i \cdot n(B) \cdot RT/V$ for strong electrolytes, where i denotes the van't Hoff factor, and MacMillan–Mayer virial expansions of the form

$$\frac{\Delta\pi}{RT} = c(B) + \mathbf{B}_2 c(B)^2 + \mathbf{B}_3 c(B)^3 + \dots \quad (13)$$

with $c(B) = n(B)/V$ and $\mathbf{B}_2, \mathbf{B}_3, \dots$ denoting virial coefficients³³. In such cases, the integral can be evaluated analytically, and explicit expressions for ΔG_{mix}^{Test} and related thermodynamic quantities can be obtained.

In the van't Hoff ideal limit, the mixing free energy of the *Test* solution can thus be expressed in terms of the osmotic work as

$$\Delta G_{mix}^{vH}(n(A), n(B)) = \Delta G_{mix}^{Final}(n(A) + n^{Dil}(A), n(B)) + \int_{V(Test)}^{V(Final)} \Delta P(V) dV \quad (14)$$

At osmotic equilibrium, $\Delta P(V) = \Delta\pi(V)$ is obtained, and the dimensionless dilution factor

$$\psi = \frac{\Delta V(Test)}{V(Test)} \quad (15)$$

is introduced, such that

$$V(Final) = V(Test) + \Delta V(Test) = (1 + \psi) \cdot V(Test) \quad (16)$$

Accordingly, the work integral is written as

$$\Delta G_{mix}^{vH}(n(A), n(B)) = \Delta G_{mix}^{Final}(n(A) + n^{Dil}(A), n(B)) + \int_{V(Test)}^{(1+\psi) \cdot V(Test)} \frac{n(B)RT}{V} dV \quad (17)$$

which yields

$$\Delta G_{mix}^{vH}(n(A), n(B)) = \Delta G_{mix}^{Final}(n(A) + n^{Dil}(A), n(B)) + n(B)RT \cdot \ln(1 + \psi) \quad (18)$$

The final, strongly osmotically diluted solution is used as a residual osmotic reference state, i.e. a state for which negligible remaining osmotic working capacity is present. To avoid confusion with the absolute mixing free energy, the *extractable* dilution work relative to the final state is therefore introduced as

$$W_{avail}(Test \rightarrow Final) \equiv \int_{V(Test)}^{V(Final)} \Delta P(V) dV \quad (19)$$

such that $W_{avail}(Final \rightarrow Final) = 0$ is obtained by definition. Along the quasi-reversible path considered here, $W_{avail}(Test \rightarrow Final) = \Delta G_{Dil}$ is obtained and therefore

$$W_{avail}(Test \rightarrow Final) = \Delta G_{mix}^{Test} - \Delta G_{mix}^{Final} \quad (20)$$

It is emphasized that W_{avail} is extracted work, i.e. path-dependent, whereas ΔG_{mix} is a state function.

For residual osmotic dilutions, a strongly diluted final state is obtained and the excess contribution is reduced. In this limit, $\Delta G_E^{Final} \approx 0$ is assumed, such that $\Delta G_{mix}^{Final} \approx \Delta G_{mix}^{Final, Ideal}$ is obtained. The ideal mixing free energy of this residual osmotic reference composition is defined as

$$\Delta G_{mix}^{ref} \equiv \Delta G_{mix}^{Final, Ideal} \quad (21)$$

Here the superscript *ref* refers to the composition of the residual osmotic reference state and the ideal-mixture approximation, and no new thermodynamic potential is introduced. Accordingly, an absolute estimate of ΔG_{mix}^{Test} is obtained as

$$\Delta G_{mix}^{Test} \approx W_{avail}(Test \rightarrow Final) + \Delta G_{mix}^{ref} \quad (22)$$

With $n_A^{Final} = n(A) + n^{Dil}(A)$ and $x_i^{Final} = n_i^{Final} / (n_A^{Final} + n(B))$

$$\Delta G_{mix}^{ref} = (n_A^{Final} + n(B)) RT \cdot (x_A^{Final} \ln x_A^{Final} + x_B^{Final} \ln x_B^{Final}) \quad (23)$$

is obtained.

In the residual-dilution limit considered here, ΔG_{mix}^{Final} is therefore replaced by ΔG_{mix}^{ref} in the van't Hoff expressions, i.e. in the regime where activity coefficients approach unity. By use of the van't Hoff relation for the *Test* volume, Eq.(18) yields

$$\Delta G_{mix}^{vH}(n(A), n(B)) = \Delta G_{mix}^{ref} + n(B)RT \cdot \ln(1 + \psi) \quad (24)$$

and since $\Delta\pi^{vH}(Test) = n(B)RT/V(Test)$ is obtained, Eq.(24) is rewritten as

$$\Delta G_{mix}^{vH}(n(A), n(B)) = \Delta G_{mix}^{ref} + \Delta\pi^{vH}(Test) \cdot V(Test) \cdot \ln(1 + \psi) \quad (25)$$

For sufficiently dilute *Test* solutions, a small osmotic pressure is obtained, the osmotic influx is limited, and hence $\psi =$



$\Delta V(Test)/V(Test) \ll 1$ is obtained, such that $\ln(1 + \psi) \approx \psi$ is obtained. In this limit,

$$\begin{aligned} \Delta G_{mix}^{vH}(Test) &\approx \Delta G_{mix}^{ref} + \Delta \pi^{vH}(Test) \cdot V(Test) \cdot \psi \\ &= \Delta G_{mix}^{ref} + \Delta \pi^{vH}(Test) \cdot \Delta V(Test) \end{aligned} \quad (26)$$

is obtained.

2.2 Riemann approximation and experimental implementation

In most practical systems, $\Delta \pi(V)$ is not available in a simple closed analytical form. The dilution in the osmotic engine is therefore implemented as a scanning, nearly reversible Kedem–Katchalsky process; please refer to Eq.(46). A sequence of counterpressures ΔP_k , with $k = 1, 2, 3 \dots$, is chosen. For each ΔP_k , the system is allowed to evolve until the volumetric flux approaches zero, $J_V \rightarrow 0$, thereby defining a quasi-equilibrium at this pressure. The solution volume in the *B*-chamber changes from V_{k-1} to V_k , and the corresponding volume increment is

$$\Delta V_{Exp,k} = V_k - V_{k-1} \quad (27)$$

In practice, the increments $\Delta V_{Exp,k}$ can be fixed mechanically by a well-defined displacement of a piston or crank mechanism, so that the volume steps are determined by the geometry of the apparatus. The braking torque, or applied load, is then adjusted until the crank is at rest and the measured pressure difference satisfies $J_V \approx 0$ at the new volume. When the crank is at rest, the counterpressure ΔP_k is read, and $\Delta V_{Exp,k}$ is obtained from simple geometrical relations. In this way, a set of paired data ($\Delta P_k, \Delta V_{Exp,k}$) is generated along a Kedem–Katchalsky, nearly reversible path from $V(Test)$ to $V(Final)$.

The continuous integral for the dilution work

$$\Delta G_{Dil} = \int_{V(Test)}^{V(Final)} \Delta P(V) dV \quad (28)$$

is then replaced by the corresponding Riemann approximation. Since ΔP_k is measured at the end of step k , i.e. at V_k , the approximation corresponds to a right-endpoint Riemann sum

$$\Delta G_{Dil} \approx \sum_k \Delta P_k \Delta V_{Exp,k} \quad \text{for} \quad \Delta P_k = \Delta P(V_k) \quad (29)$$

Substitution into the operational expression for the mixing free energy in the *Test* state yields the experimentally accessible approximation

$$\begin{aligned} \Delta G_{mix}^{Test}(n(A), n(B)) &\approx \Delta G_{mix}^{Final}(n(A) + n^{Dil}(A), n(B)) \\ &\quad + \sum_k \Delta P_k \Delta V_{Exp,k} \end{aligned} \quad (30)$$

For residual osmotic dilutions, the final, osmotically diluted solution is used as a residual osmotic reference state for which the remaining osmotic working capacity is negligible and activity coefficients approach unity. In this limit one may approximate

$\Delta G_{mix}^{Final} \approx \Delta G_{mix}^{ref}$, refer to Eq.(21), and therefore

$$\Delta G_{mix}^{Test}(n(A), n(B)) \approx \Delta G_{mix}^{ref} + \sum_k \Delta P_k \Delta V_{Exp,k} \quad (31)$$

The quality of the Riemann approximation depends on the magnitudes of the steps $\Delta V_{Exp,k}$ and the resulting changes in ΔP_k . If the steps become too large, the path becomes both numerically coarse and thermodynamically irreversible, with non-negligible entropy production in each step. In practice, small and mechanically well-controlled volume steps and a sensitive mechanical gearing are therefore advantageous for maintaining $J_V \approx 0$ and a nearly reversible path. As a consistency check, $V(Final)$ can be obtained either from direct volume readout or as $V(Final) = V(Test) + \sum_k \Delta V_{Exp,k}$.

An electric brake engine or generator can be employed to provide the required braking torque on the shaft of the osmotic engine. For each step k , the braking current is adjusted until the volumetric flux vanishes and the crank is at rest, $J_V \rightarrow 0$. The corresponding counterpressure ΔP_k is then determined, the volume increment $\Delta V_{Exp,k}$ is obtained from the geometry of the apparatus, and the final volume $V(Final)$ of the diluted solution is measured or inferred from the step sum. In this manner, all quantities required to evaluate the Riemann approximation and thus $\Delta G_{mix}^{Test}(n(A), n(B))$ for a given practically incompressible binary solution are obtained experimentally.

Moreover, if the paired values of the braking current I_{Brake} and the corresponding electromotive force U_{Brake} are recorded for each step k , the electric brake engine can simultaneously be operated as a generator. In that case, the PRO dilution step and the corresponding RO concentration step may be viewed as two branches of a closed electro-osmotic cycle, formally analogous to cyclic voltammetry. This opens the possibility of representing the osmotic engine as a cyclic voltammogram in the (U_{Brake}, I_{Brake}) -plane. In the quasi-reversible limit and neglecting electrical losses, or after calibration, the electrical work delivered over a closed electro-osmotic cycle equals the osmotic work, so that the area enclosed by the loop in the (U_{Brake}, I_{Brake}) -plane is proportional to W_{avail} and hence to $-\Delta G_{sys}^{Dil}$. Such a representation may be particularly appealing when the solute *B* undergoes conformation changes or other structural rearrangements during dilution; in such systems, modifications of the loop shape or hysteresis could provide an experimental handle on slow, conformation-dependent contributions to $\Delta G_{mix}^{Test}(n(A), n(B))$.

2.3 The Conceptual Osmotic Engine

An idealized, conceptual osmotic engine is considered, consisting of two reservoirs separated by a semipermeable membrane. The engine comprises at least three cylinders, each with a fixed volume V_C , all connected to a common crankshaft. At least three cylinders are used to ensure continuous torque delivery on the crankshaft while one cylinder is in the working stage.

The engine operates through an *osmotic* cycle with three distinct stages. In the filling stage, *F*, a cylinder is partially charged with a volume αV_C of the *Test* solution with density $\rho(Test)$. During the working stage, *W*, this volume is osmotically diluted while



mechanical work, W , is delivered to the crankshaft and thereby to the surroundings. In the exhaust stage, E , the osmotically diluted $Test$ solution with density $\tilde{\rho}(Test)$ is discharged, and the cycle is repeated. The cylinder sequence can thus be summarized as (F, W, E) , with only one cylinder in the working stage at any given time.

The pressure contribution is associated with the actual osmotic pressure of the $Test$ solution, and the change in cylinder volume, ΔV_C , is confined to

$$\Delta V_C = (1 - \alpha) \cdot V_C \quad (32)$$

where α denotes a filling fraction between zero and one.

The first reservoir is filled with the pure solvent A , whereas the second reservoir contains solute B dissolved in A to form a non-reactive binary $Test$ solution. Initially, the $Test$ solution occupies the volume αV_C within the working cylinder. The system is characterized by an observed osmotic pressure difference $\Delta\pi^{obs}(Test)$ relative to the pure solvent. Operationally, in the quasi-reversible limit, $\Delta\pi^{obs}(V)$ denotes the counterpressure ΔP required to reach a quasi-equilibrium condition $J_V \rightarrow 0$ at the corresponding cylinder volume V , so that $\Delta\pi^{obs}(V) = \Delta P(V)$ along the quasi-reversible path.

A hypothetical free-expansion equilibrium volume V_C^{eq} is defined as the volume that would be attained if the piston were allowed to expand freely against the surroundings, i.e. with vanishing applied counterpressure. In addition, a critical stall osmotic pressure $\Delta\pi_{cri}^{obs}(Test)$ is defined as the counterpressure at which the piston cannot advance further within the admissible expansion range between αV_C and V_C under quasi-equilibrium conditions $J_V \rightarrow 0$, i.e. the limiting mechanical equilibrium where the net driving pressure across the membrane is balanced by the applied counterpressure and the piston velocity vanishes.

Moreover, it is observed, see Supplementary Material: The Concept of the Osmotic Engine, that $\Delta\pi_{cri}^{obs}(Test) \rightarrow \Delta\pi^{obs}(Test)^{Dil}$, the residual osmotic pressure, as $\alpha \rightarrow 1$. For a more detailed description of the osmotic engine, reference is made to Fig.(4) and Fig.(5), as well as to the supplemental material: The Concept of the Osmotic Engine.

2.4 Non-Ideal Solutions

At osmotic equilibrium, the corresponding change in cylinder volume of the $Test$ solution, $\Delta V_C^{eq}(Test)$, is given by

$$\Delta V_C^{eq}(Test) = V_C^{eq}(Test) - \alpha V_C \quad (33)$$

A convenient determination of $V_C^{eq}(Test)$ can be achieved by recirculating the exhaust volume back into the $Test$ solution, so that the solution is gradually diluted. At a certain dilution step, the critical osmotic pressure, $\Delta\pi_{cri}^{obs}(Test)$, becomes equal to the hydrostatic pressure associated with a column height between the initial cylinder volume αV_C and the final cylinder volume V_C , at which point the engine stalls.

When the system is brought to osmotic equilibrium, the resulting membrane flux is reduced to $J_V \rightarrow 0$, and the actual osmotic pressure is obtained from the measured counterpressure, ΔP_k , in the cylinder compartment. This measured pressure is paired with

the corresponding volume increment $\Delta V_{Exp,k}$, determined from straightforward analytical geometry. Accordingly, using Eq.(30), the measured Gibbs free energy of mixing can be expressed in the residual-dilution approximation as

$$\Delta G_{mix}^{Test}(n(A), n(B)) \approx \Delta G_{mix}^{ref} + \sum_k \Delta P_k \cdot \Delta V_{Exp,k} \quad (34)$$

Here the cumulative experimentally imposed volume change is identified with the equilibrium expansion

$$\sum_k \Delta V_{Exp,k} = \Delta V_C^{eq}(Test) \quad (35)$$

and therefore

$$V_{Final} = V(Test) + \Delta V_C^{eq}(Test) = (1 + \psi) \cdot V(Test) \quad (36)$$

If the residual-dilution approximation is not invoked, the corresponding expression is instead written as

$$\Delta G_{mix}^{Test}(n(A), n(B)) \approx \Delta G_{mix}^{Final}(n(A) + n^{Dil}(A), n(B)) + \sum_k \Delta P_k \cdot \Delta V_{Exp,k} \quad (37)$$

which reduces to Eq.(34) when $\Delta G_{mix}^{Final} \approx \Delta G_{mix}^{ref}$, i.e. in the strongly diluted regime where the excess contribution becomes negligible.

From the above, it follows that when $V_C^{eq}(Test) < V_C$, the operational cylinder is brought to rest, the osmotic cycle is interrupted, and the engine comes to a complete stop. It should also be noted that $V_C^{eq}(Test)$ approaches V_C arbitrarily closely as α approaches unity.

The dilution of the $Test$ solution up to the point at which the engine stalls, and thereby the determination of the critical dilution factor ψ , can be carried out in a straightforward manner with the osmotic engine. This dilution factor is closely related to the hypothetical osmotic equilibrium volume $V_C^{eq}(Test)$ of the $Test$ solution and the cylinder volume V_C . Since

$$\Delta V_C^{eq}(Test) = V_C^{eq}(Test) - \alpha V_C \quad (38)$$

it follows that

$$V_C^{eq}(Test) = \alpha V_C + \Delta V_C^{eq}(Test) \quad (39)$$

If the critical dilution factor is defined relative to the initially loaded $Test$ volume

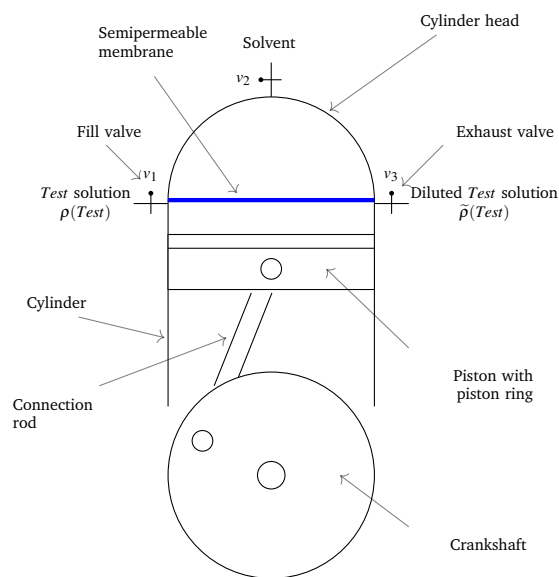
$$\psi \equiv \frac{\Delta V_C^{eq}(Test)}{\alpha V_C} \quad (40)$$

then

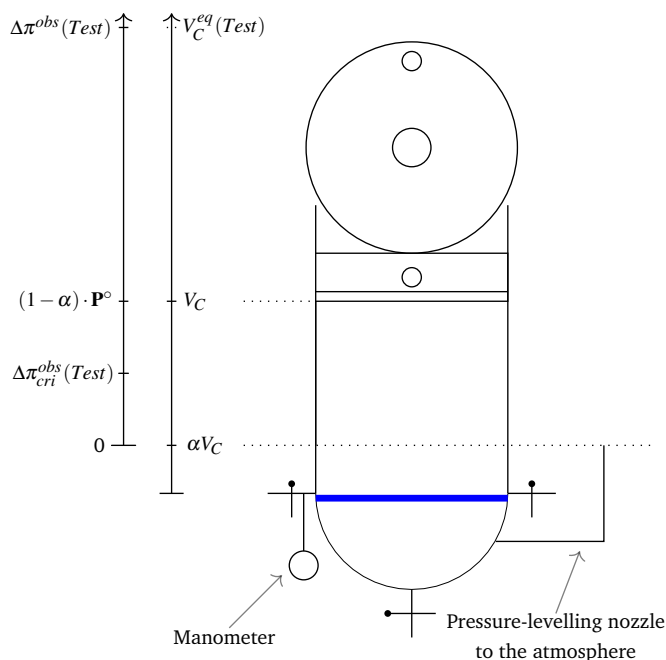
$$V_C^{eq}(Test) = (1 + \psi) \cdot \alpha V_C \quad (41)$$

For a more detailed description, the reader is referred to Fig.(9) and to the supplemental material: Calibrating the Conceptual Osmotic Engine. Within the present framework of non-reactive binary solutions and the quasi-reversible operating conditions described above, the Gibbs free mixing energy at constant temperature is then evaluated from Eq.(34).





(a) Osmotic cylinder prepared to harness the osmotic energy potential in the *Test* solution.



(b) An osmotic cylinder configured as an osmometer.

Fig. (4) (a) In this embodiment, it is preferable that the membrane area, A_M , exceeds the piston cross-section, A_C . (b) By adding a pressure-levelling nozzle, the condition $(\Delta P_k)_{J_V=0}$ can be measured.

In the dilution scenario where the engine stalls, the osmotic pressure of the diluted *Test* solution is referred to as the critical osmotic pressure, $\Delta\pi_{cri}^{obs}(Test)$. In this limiting state, the piston cannot advance further within the admissible expansion range, and the system is at quasi-equilibrium with $J_V \rightarrow 0$. The piston is positioned between the least osmotically diluted liquid column, $\alpha V_C/A_C$, and the most osmotically diluted, V_C/A_C , where A_C represents the cross-sectional area.

As shown in the supplemental material: Calibrating the Conceptual Osmotic Engine, under the present geometric calibration one obtains

$$0 \leq \Delta\pi_{cri}^{obs}(Test) \leq (1 - \alpha) \cdot \mathbf{P}^\circ \quad (42)$$

where $\mathbf{P}^\circ = 1$ bar is the standard pressure.

2.4.1 A Practical Application

The capability to estimate the Gibbs free energy of mixing through Eq.(34) for a real binary solution carries significant scientific and technical interest. In particular, once the absolute mixing free energy of the *Test* composition, $\Delta G_{mix}^{Test}(n(A), n(B))$, has been estimated through the measured osmotic work together with the residual-reference correction introduced above, the corresponding Gibbs free excess energy, ΔG_E , can also be estimated. This is of practical relevance, for example, in the assessment of parameters such as the Flory–Huggins interaction parameter, χ , for a real binary polymer–solvent solution^{8,9}. For this special case, the general notation is specialized such that the solvent *A* is identified with *S* and the solute *B* is identified with the polymer *P*. The figures below illustrate conceptual osmotic-engine configurations relevant to such practical implementations.

In the present context, $\Delta G_{mix}^{Test}(n(A), n(B))$ denotes the absolute mixing free energy of the *Test* composition, whereas $\Delta G_{mix}^{ideal, FH}(n(A), n(B))$ denotes the corresponding Flory–Huggins ideal (combinatorial) mixing contribution evaluated for the same composition and within the same reference frame. Under this condition, the excess energy of the *Test* solution is estimated as

$$\Delta G_E = \Delta G_{mix}^{Test}(n(S), n(P)) - \Delta G_{mix}^{ideal, FH}(n(S), n(P)) \quad (43)$$

Thus, for a real binary polymer–solvent solution treated within the Flory–Huggins framework, the experimentally estimated $\Delta G_{mix}^{Test}(n(A), n(B))$ may be combined with the corresponding ideal Flory–Huggins mixing term to obtain ΔG_E , and thereby the interaction parameter χ . In this polymer–solvent specialization, the Flory–Huggins quantities are written as

$$\chi = \frac{\Delta G_E}{\mathbf{RT} \cdot n(S) \cdot \phi(P)} \quad (44)$$

where

$$\frac{\Delta G_{mix}^{ideal, FH}}{\mathbf{RT}} = n(S) \cdot \ln \phi(S) + n(P) \cdot \ln \phi(P) \quad (45)$$

Here, $n(S)$ denotes the number of moles of solvent, and $n(P)$ denotes the number of moles of polymer chains in the Flory–Huggins reference description. Likewise, $\phi(S)$ and $\phi(P)$ denote the volume fractions of solvent and polymer, respectively (not to be confused with the osmotic coefficient of the *Test* solution). Finally, $\chi = \chi(\phi(P), T)$ denotes the Flory–Huggins interaction parameter between solvent and polymer.



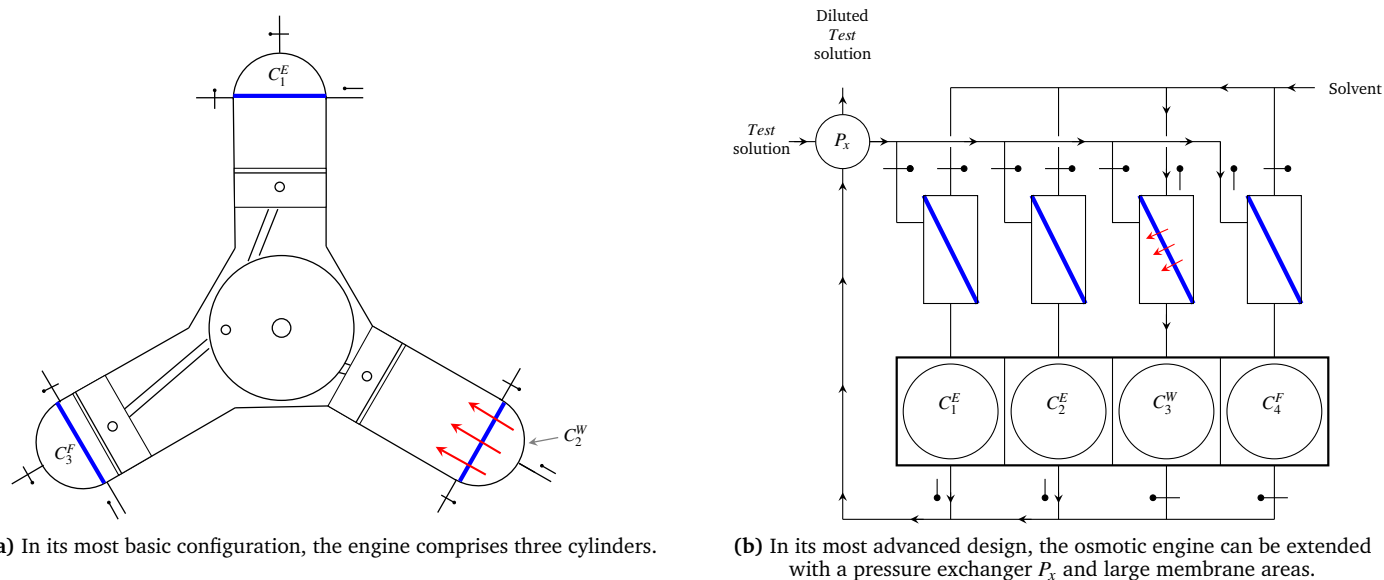


Fig.(5) (a) Star engine configuration: When $\alpha = 1/3$ the engine has three cylinders. (b) Inline engine viewed from below: When $\alpha = 1/2$ it has four cylinders. The pistons move downward as in Fig.(4(b)) when the system acts as a membrane osmometer. For clarity, pressure nozzles, manometer and crankshaft are omitted.

2.5 Van't Hoff Ideal Solutions

The transport of liquid through a membrane is a non-equilibrium process and is therefore most naturally described within the framework of irreversible thermodynamics. Following the seminal developments by Kedem and Katchalsky^{12–14} and subsequent contributions^{15–19}, membrane transport in binary systems is commonly modeled in terms of coupled driving forces. In the near-equilibrium regime, the volumetric flux, J_V , is governed by the competition between the osmotic pressure difference, $\Delta\pi$, and the hydrostatic pressure difference, ΔP . In the present work, J_V is considered under PRO conditions; accordingly, the sign of J_V follows the PRO flux direction implied by the adopted sign convention for ΔP and $\Delta\pi$, and may therefore appear reversed relative to the convention often used for RO. For a semipermeable membrane separating two non-reacting components - e.g., a solvent A and a solute B — the volume flux may be written as

$$J_V = L_p \cdot (\sigma\Delta\pi - \Delta P) \quad \text{for} \quad 0 \leq \Delta P \leq \sigma\Delta\pi \quad (46)$$

where L_p is the hydraulic permeability of the membrane. The parameter σ is the solute reflection coefficient and quantifies membrane selectivity: if $\sigma = 0$, the membrane offers no selectivity and B can pass freely; if $\sigma = 1$, the membrane is ideally semipermeable and B is completely rejected. For macromolecular solutes, a typical expectation is $\sigma \approx 1$ when using a reverse-osmosis-type membrane, i.e. the solute is effectively retained. In the following, the membrane is assumed to be perfectly semipermeable, which precisely corresponds to setting $\sigma = 1$. Because J_V is the total volume flow per membrane surface area, A_M , it follows that

$$\frac{1}{A_M} \frac{\partial V}{\partial t} = J_V \quad (47)$$

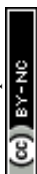
where V represents the volume of the compartment where the solute is dissolved. It follows from Eq.(46) that the process power, $\partial W / \partial t$, is expressed as

$$\frac{\partial W}{\partial t} = A_M L_p (\Delta\pi - \Delta P) \Delta P \quad (48)$$

where W represents the pressure-volume work executed during the spontaneous transport process. Notably, $\partial W / \partial t$, the power transferred to the crankshaft, $\tau_E \cdot \omega_E$, can be estimated from the chosen torque, τ_E , and the measured crank angular velocity, ω_E , on the crankshaft of the engine. Hence, depending on the chosen τ_E , the engine operates between two distinct modes: Reverse Osmosis if $\omega_E < 0$ and Pressure Retarded Osmosis if $0 \leq \omega_E \leq \max(\omega_E)$.

In the subsequent section, the van't Hoff relation for the osmotic pressure difference, Eq.(12), is adopted as an initial approximation. If this ideal assumption proves inadequate, the framework may be refined through the incorporation of higher-order corrections via MacMillan–Mayer virial coefficients³³, which are expected to constitute the dominant contribution in a more realistic treatment. Accordingly, Eq.(59) and Eq.(71) are presented as the leading-order results of a first-order virial correction. Furthermore, the osmotic engine may be realized either as a concrete measurement device, for example to determine the Flory–Huggins interaction parameter in Eq.(44), or as an idealized engine for theoretical investigations, analogous to the Carnot heat engine.

The osmotic engine consists of a cylindrical configuration, as depicted in Fig.(4(b)). The operation of the piston involves oscillating between three pivotal positions: initially resting in the vacant state denoted as P_1 and transitioning to the subsequent state containing a pristine, undiluted *Test* solution labeled as P_2 . Upon reaching position P_2 , solvent permeation through a semiper-



meable membrane is initiated, engaging in PV -work until the system arrives at the concluding position P_3 , thereby instigating the discharge process. Subsequently, the cylinder returns to position P_1 to commence the cycle anew. Throughout the working stroke, the solute amount $n_C(B)$ in the initial volume αV_C at P_2 remains constant. A decrease in the actual osmotic pressure, denoted as $[\Delta\pi(T_{est})]$, occurs within the range from $\Delta\pi^{vH}(T_{est})$ to $\alpha\Delta\pi^{vH}(T_{est})$. This decrease is due to an increase in solvent volume, as described by Eq.(12), where α represents the initial fraction of the cylinder volume V_C filled with undiluted T_{est} solution. The change in Gibbs free energy of mixing is calculated by defining ΔP as directly proportional to $[\Delta\pi(T_{est})]$, with a constant k , $0 \leq k < 1$, along the dynamic branch. The limiting case $k \rightarrow 1^-$ corresponds to the quasi-static stall limit, whereas $k = 0$ corresponds to a static membrane-osmometer-like configuration.

The volume is arbitrarily set at αV_C at the start of the working stroke, causing an increase in volume and a decrease in actual osmotic pressure. Hence, it follows from Eq.(46) and Eq.(47) that

$$\begin{aligned} \frac{\partial V^{(0)}}{\partial t} &= A_M L_p \cdot \left([\Delta\pi(T_{est})] - \Delta P \right) \\ &= A_M L_p \cdot n_C(B) \mathbf{RT} \cdot (1-k) \frac{1}{V^{(0)}} \end{aligned} \quad (49)$$

At the time $t = 0$ the piston initiates its working stroke, $V^{(0)}(0) = \alpha V_C$, and thus Eq.(49) is integrated to

$$V^{(0)}(t) = \sqrt{2A_M L_p \cdot n_C(B) \mathbf{RT} \cdot (1-k)t + (\alpha V_C)^2} \quad (50)$$

The time needed to complete the working stroke, $\Gamma_C^{(0)}$, is identified through Eq.(50) as

$$\begin{aligned} V^{(0)}(t = \Gamma_C^{(0)}) = V_C &= \sqrt{2A_M L_p \cdot n_C(B) \mathbf{RT} \cdot (1-k)\Gamma_C^{(0)} + (\alpha V_C)^2} \Rightarrow \\ \Gamma_C^{(0)} &= \frac{(1-\alpha^2)V_C^2}{2A_M L_p \cdot n_C(B) \mathbf{RT} \cdot (1-k)} \end{aligned} \quad (51)$$

The time needed to make the engine complete one cycle, $\Gamma_E^{(0)}$, is identified as the product of the number of engine pistons, $N = \frac{2}{1-\alpha}$, and the time needed for one piston to complete the working stroke, $\Gamma_C^{(0)}$, i.e.

$$\begin{aligned} \Gamma_E^{(0)} &= N \cdot \Gamma_C^{(0)} \\ &= \frac{(1+\alpha)V_C^2}{A_M L_p \cdot n_C(B) \mathbf{RT} \cdot (1-k)} \end{aligned} \quad (52)$$

The angular velocity, $\omega_E^{(0)}$, of the crankshaft is expressed as the reciprocal value of $\Gamma_E^{(0)}$ multiplied by 2π

$$\begin{aligned} \omega_E^{(0)} &= \frac{2\pi}{\Gamma_E^{(0)}} \\ &= 2\pi \cdot \frac{A_M L_p \cdot n_C(B) \mathbf{RT} \cdot (1-k)}{(1+\alpha)V_C^2} \end{aligned} \quad (53)$$

where π is the mathematical constant, not to be confused with osmotic pressure. Assuming mechanical losses in the engine to be negligible, the power measured on the crankshaft is equal to the product of its angular velocity, ω_E , and the applied torque, τ_E , due to only one piston being in its working stroke at any given time. Hence, combining Eq.(48) and Eq.(53) yields

$$\begin{aligned} \frac{\partial W}{\partial t} &= \omega_E^{(0)} \cdot \tau_E^{(0)} \\ &= \left(2\pi \cdot \frac{A_M L_p \cdot n_C(B) \mathbf{RT} \cdot (1-k)}{(1+\alpha)V_C^2} \right) \cdot \tau_E^{(0)} \Rightarrow \\ \tau_E^{(0)} &= \frac{n_C(B) \mathbf{RT}}{2\pi} \cdot (1+\alpha)k \cdot \frac{V_C^2}{2A_M L_p \cdot n_C(B) \mathbf{RT} \cdot (1-k)t + \alpha^2 V_C^2} \end{aligned} \quad (54)$$

Equation (54) is valid along the dynamic branch $0 \leq k < 1$. The stall torque is therefore obtained as a limiting value, not by direct substitution at $k = 1$, namely

$$\max \{ \tau_E^{(0)} \} \equiv \lim_{k \rightarrow 1^-} \tau_E^{(0)} \quad (55)$$

Taking this limit in Eq.(54) yields

$$\begin{aligned} \max \{ \tau_E^{(0)} \} &= \frac{1}{2\pi} \cdot n_C(B) \mathbf{RT} \cdot \left(\frac{1+\alpha}{\alpha^2} \right) \\ &= \frac{1}{2\pi} \cdot \left(\frac{n_C(B) \mathbf{RT}}{\alpha V_C} \right) \cdot \alpha V_C \cdot \left(\frac{1+\alpha}{\alpha^2} \right) \\ &= \frac{1}{2\pi} \cdot \left(\frac{1+\alpha}{\alpha} \right) \cdot \Delta\pi^{vH}(T_{est}) \cdot V_C \end{aligned} \quad (56)$$

from which it follows that the van't Hoff ideal osmotic pressure of the T_{est} solution is expressed as

$$\Delta\pi^{vH}(T_{est}) = 2\pi \cdot \left(\frac{\alpha}{1+\alpha} \right) \cdot \left(\frac{\max \{ \tau_E^{(0)} \}}{V_C} \right) \quad (57)$$

and like this

$$\theta^{obs}(T_{est}) = 2\pi \cdot \left(\frac{\alpha}{1+\alpha} \right) \cdot \left(\frac{\max \{ \tau_E^{obs} \}}{V_C \cdot \Delta\pi^{vH}(T_{est})} \right) \quad (58)$$

where $\theta^{obs}(T_{est})$ and $\max(\tau_E^{obs})$ are the observed osmotic coefficient and the stalling torque of the T_{est} solution.

In the present reference convention, the following expression represents the *absolute* van't Hoff estimate of the mixing free energy, obtained by adding the residual reference contribution to the extractable osmotic work term. When Eq.(25) and Eq.(57) are combined, the van't Hoff contribution to the mixing free energy is obtained as

$$\begin{aligned} \Delta G_{mix}^{vH(0)} &= \Delta G_{mix}^{ref} \\ &+ 2\pi \cdot \left(\frac{\alpha}{1+\alpha} \right) \cdot \left(\frac{\max \{ \tau_E^{(0)} \}}{V_C} \right) \cdot V(T_{est}) \cdot \ln(1+\psi) \end{aligned} \quad (59)$$



where ψ is determined either by the dilution procedure outlined in the supplemental material: Calibrating the Conceptual Osmotic Engine, or, when volume additivity is assumed between the pure solvent A and the $Test$ solution, by applying Eq.(S28) or Eq.(S29). By construction, inserting Eq.(56) and Eq.(57) into Eq.(59) recovers Eq.(25) in the van't Hoff limit. The latter is regarded as the zero-order perturbation for a real binary solution, i.e. $\Delta G_{mix}^{(0)} = \Delta G_{mix}^{vH}$. To deduce, for example, the first-order perturbation, the MacMillan–Mayer expression including the second virial coefficient is substituted into the Kedem–Katchalsky relation in Eq.(46), and Eq.(49) is re-integrated. This procedure is employed to extract the contribution of order ε in the perturbation expansion, while all terms of order ε^2 and higher are systematically neglected.

2.5.1 A Theoretical Perspective

The excess Gibbs energy, ΔG_E , is defined as the difference between the actual Gibbs energy of mixing, ΔG_{mix} , and the ideal Gibbs energy of mixing,

$$\Delta G_{mix}^{ideal} = \mathbf{RT} \left(n(A) \ln x_A + n(B) \ln x_B \right) \quad (60)$$

Hence,

$$\Delta G_E = \Delta G_{mix} - \mathbf{RT} \left(n(A) \ln x_A + n(B) \ln x_B \right) \quad (61)$$

thereby adopting Raoult's reference frame, i.e. ideal behaviour of each component in its pure state, as the zero point for the subsequent mathematical treatment. The activity coefficient of component B , γ_B^R , is defined via the partial derivative of ΔG_E with respect to $n(B)$, at constant T , P , and $n(A)$

$$\ln \gamma_B^R = \frac{1}{\mathbf{RT}} \left(\frac{\partial \Delta G_E}{\partial n(B)} \right) \quad (62)$$

Upon substituting Eq.(61) and noting that, for constant $n(A)$ with $n_T = n(A) + n(B)$

$$\frac{\partial}{\partial n(B)} \left(n(A) \ln x_A + n(B) \ln x_B \right) = \ln x_B \quad (63)$$

it follows that

$$\frac{\partial \Delta G_{mix}^{ideal}}{\partial n(B)} = \mathbf{RT} \ln x_B \quad (64)$$

Inserting Eq.(64) into Eq.(62) yields

$$\ln \gamma_B^R = \frac{1}{\mathbf{RT}} \left(\frac{\partial \Delta G_{mix}}{\partial n(B)} \right) - \ln x_B \quad (65)$$

where $-\ln x_B$ is the ideal Raoult-reference term, and deviations from ideality are captured by the remaining term.

According to Eq.(25), for van't Hoff ideal solutions, it follows

$$\Delta G_{mix}^{vH}(n(A), n(B)) = \Delta G_{mix}^{ref} + \Delta \pi^{vH}(Test) \cdot V(Test) \cdot \ln(1 + \psi) \quad (66)$$

However, even if ΔG_{mix}^{ref} becomes numerically small in the dilute limit, its composition derivative need not be small. Therefore, $\Delta G_{mix}^{ref} \approx 0$ does not in itself imply that $\partial \Delta G_{mix}^{ref} / \partial n(B) \approx 0$.

With this choice, Eq.(65) can be rewritten as

$$\ln \gamma_B^R = \frac{1}{\mathbf{RT}} \frac{\partial}{\partial n(B)} \left(\Delta G_{mix}^{ref} + \Delta \pi^{vH}(Test) \cdot V(Test) \cdot \ln(1 + \psi) \right) - \ln x_B$$

$$= \frac{1}{\mathbf{RT}} \frac{\partial \Delta G_{mix}^{ref}}{\partial n(B)} + \frac{1}{\mathbf{RT}} \frac{\partial}{\partial n(B)} \left(n(B) \mathbf{RT} \ln(1 + \psi) \right) - \ln x_B \quad (67)$$

In accordance with Eq.(S77) it follows that

$$\frac{1}{\mathbf{RT}} \frac{\partial \Delta G_{mix}^{ref}}{\partial n(B)} \approx \ln x_B \quad (68)$$

for small values of $n(B)$ in the van't Hoff ideal case. Under this approximation,

$$\begin{aligned} \ln \gamma_B^R &= \frac{1}{\mathbf{RT}} \frac{\partial}{\partial n(B)} \left(\Delta G_{mix}^{ref} + \Delta \pi^{vH}(Test) \cdot V(Test) \cdot \ln(1 + \psi) \right) - \ln x_B \\ &= \frac{\partial}{\partial n(B)} \left(n(B) \ln(1 + \psi) \right) \end{aligned} \quad (69)$$

This expression also holds within the working cylinder of a conceptual osmotic engine with a large number of cylinders. In the supplementary material: Calibrating the Conceptual Osmotic Engine, it is shown that $n_C(B) \partial \psi / \partial n_C(B) = n(B) \partial \psi / \partial n(B)$, yielding

$$\begin{aligned} n(B) \frac{\partial \psi}{\partial n(B)} &= \\ \frac{\Delta \hat{\pi}(Test)}{\Delta \hat{\pi}(Test) + 1} \left(\Delta \hat{\pi}(Test) + 1 + m(B) \cdot \mathbf{M}(B) \cdot \left(\frac{\rho^*(A) - \rho^*(B)}{\rho^*(B)} \right) \right) \end{aligned} \quad (70)$$

according to Eq.(S43), and where $m(B)$ is the solute molality. This result is derived under the additional assumption of volume additivity between A and B . Note that the dilute-solution approximation $\psi \approx \Delta \hat{\pi}(Test)$ is used in the derivation of Eq.(70). Hence, it follows

$$\begin{aligned} \ln \gamma_B^R &= \ln \left(1 + \Delta \hat{\pi}(Test) \right) \\ &+ \frac{\Delta \hat{\pi}(Test)}{\left(1 + \Delta \hat{\pi}(Test) \right)^2} \left(\Delta \hat{\pi}(Test) + 1 + m(B) \cdot \mathbf{M}(B) \cdot \left(\frac{\rho^*(A) - \rho^*(B)}{\rho^*(B)} \right) \right) \end{aligned} \quad (71)$$

The Raoult-law activity coefficient, γ_B^R , may be identified for a volatile component B through its fugacity $f_B(x_B, T, P)$ according to

$$f_B(x_B, T, P) = x_B \gamma_B^R \mathbf{P}_B^* \Rightarrow \mathbf{P}_B^* = \lim_{x_B \rightarrow 1} \left(\frac{f_B(x_B, T, P)}{x_B} \right) \quad (72)$$

where \mathbf{P}_B^* is the vapour pressure of pure B . Consequently, $\gamma_B^R \rightarrow 1$ as $x_B \rightarrow 1$.

For completeness, one may also introduce a Henry-scale activity coefficient, γ_B^H , through

$$f_B(x_B, T, P) = x_B \gamma_B^H \mathbf{H}_B \Rightarrow \mathbf{H}_B = \lim_{x_B \rightarrow 0} \left(\frac{f_B(x_B, T, P)}{x_B} \right) \quad (73)$$

where \mathbf{H}_B is Henry's constant. Combining Eq.(72) and Eq.(73)



gives

$$\gamma_B^R = \gamma_B^H \cdot \left(\frac{\mathbf{H}_B}{\mathbf{P}_B^*} \right) \quad (74)$$

which corresponds to a change of standard state between Raoult and Henry conventions. In the present work, however, Eq.(71) is derived on the Raoult scale. Therefore, Eq.(71) should be interpreted as a Raoult-reference result. A further Henry-normalized interpretation would require an additional asymptotic analysis to demonstrate explicitly that $\gamma_B^H \rightarrow 1$ as $x_B \rightarrow 0$, which is not established here.

Perspective and limitations: While Eq.(71) is algebraically consistent within the present osmotic-engine framework, quantitative use of the resulting estimate of $\ln \gamma_B^R$ requires caution. The dominant sensitivity enters through the treatment of the solution volume in the dilute limit. In particular, the auxiliary assumption of volume additivity, $V(\text{Test}) = n(A)\bar{V}_A + n(B)\bar{V}_B$, is only asymptotically justified as $n(B) \rightarrow 0$ if the partial molar solute volume approaches its infinite-dilution value \bar{V}_B^∞ in a controlled manner. Replacing this limiting partial molar quantity by the pure-component molar volume, $\bar{V}_B^\infty \approx \mathbf{V}_B^* = \mathbf{M}(\mathbf{B})/\rho(\mathbf{B})$, is a crude closure: For many solutes, \bar{V}_B^∞ differs substantially from \mathbf{V}_B^* because solvation/structuring effects, compressibility differences, and excess mixing volumes shift the effective volumetric contribution of a single dissolved molecule away from its pure-liquid value. Consequently, the density-based term in Eq.(70) and therefore the corrective contribution in Eq.(71) can be biased in both magnitude and sign for specific solute–solvent pairs.

A second, independent source of fragility is the dilute-solution approximation $\psi \approx \Delta\hat{\pi}(\text{Test})$ used in deriving Eq.(70). This step suppresses higher-order coupling between osmotic work capacity and composition-dependent volume changes, i.e. it effectively truncates the concentration expansion at a stage where real solutions may already exhibit measurable non-ideality in $\Delta\pi$ and in the partial molar volumes. In the same spirit, the van't Hoff ideal input, Eq.(66), should be read as a conceptual limiting case rather than a quantitative model for arbitrary dilute mixtures, since real systems may deviate from van't Hoff behaviour due to association, specific interactions, or, for electrolytes, non-colligative contributions that are not captured by the present closure.

By analogy with Carnot's ideal heat engine, the osmotic-engine construct remains useful as a unifying conceptual scaffold: One may envisage successive perturbation expansions of $\ln \gamma_B^R$ in powers of concentration, provided that volumetric inputs are treated in a controlled manner, e.g. via \bar{V}_B^∞ rather than \mathbf{V}_B^* . As a natural next step toward a more robust dilute-theory closure, one can re-derive an Eq.(71)-type relation using a MacMillan–Mayer virial description of the osmotic pressure,

$$\Delta\pi(\text{Test}) = \frac{n(B)RT}{V(\text{Test})} + \varepsilon RT \mathbf{B}_2 \left(\frac{n(B)}{V(\text{Test})} \right)^2 + \mathcal{O}(\varepsilon^2) \quad (75)$$

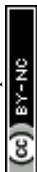
where \mathbf{B}_2 is the second osmotic virial coefficient of the solute B dissolved in the solvent A . In this view, higher-order terms admit a molecular-interaction interpretation, promoting measurable virial coefficients and infinite-dilution partial molar volumes to controlled inputs for activity predictions in complex solutions.

3 Summary

A membrane-thermodynamic framework, termed the osmotic engine, is developed to determine the excess Gibbs free energy of mixing, ΔG_E , in real binary solutions from measurable osmotic and volumetric responses under pressure-retarded osmosis, PRO, conditions. In the van't Hoff limit, the theory provides a leading-order colligative perspective linking the Henry-reference activity coefficient at infinite dilution to the dimensionless osmotic pressure, while clarifying its sensitivity to volume assumptions and reference-state regularisation. The framework extends to multi-component mixtures containing electrolytes and non-electrolytes, captures solvent-dependent shifts in reaction equilibria, and supports routine estimation of the Flory–Huggins solvent–polymer interaction parameter.

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Nomenclature

A_C	Cylindric cross sectional area, [m^2].
A_M	Membrane surface area, [m^2].
α	Fraction of cylinder volume in osmotic engine, [-].
C_n	The n^{th} cylinder, $n \in \{1, 2, 3, \dots\}$, [-].
E_n	The n^{th} exhaust stroke, [-].
F_n	The n^{th} filling stroke, [-].
W_n	The n^{th} working stroke, [-].
ΔG	The change of the Gibbs Free Energy, [J].
ΔG_{mix}	The change of the Gibbs Free Energy of mixing, [J].
$\Delta G_{\text{mix}}^{\text{prod}}$	The change of the Gibbs Free Energy of mixing of reaction products, [J].
$\Delta G_{\text{mix}}^{\text{react}}$	The change of the Gibbs Free Energy of mixing of reactants, [J].
$\Delta \bar{G}_{\text{mix}}$	The change of the partial molar Gibbs Free Energy of mixing, [$J \cdot \text{mol}^{-1}$].
ΔG_{rxn}	The change of the Gibbs Free Energy of reaction, [J].
ΔG_{Excess}	The change of the Gibbs Free Excess Energy, [J].
g	The gravitational constant, [$9.82 \dots m \cdot s^{-2}$].
Γ_C	The time needed for one single cylinder to perform the working stroke, [s].
Γ_E	The time needed for one engine revolution, [s].
Δh	The travel part between the cylinder volumes αV_C and V_C , [m].
ι	The van't Hoff factor, [-].
J_V	The volume flow across the semipermeable membrane, [$m^3 \cdot m^{-2} \cdot s^{-1}$].
k	The load on the crank shaft, [-].
K_a	The chemical affinity constant for an arbitrary addition reaction, [M^{-1}].
L_P	The solvent permeability coefficient across the semipermeable membrane, [$m^3 \cdot N^{-1} \cdot s^{-1}$].
μ°	The standard chemical potential of formation, [$J \cdot \text{mol}^{-1}$].
n	Amount of substance, [mol].
N	The number of engine cylinders, [-].
ω_E	The crank angular velocity, [$\text{rad} \cdot s^{-1}$].
ΔP	The actual pressure difference of the working cylinder under osmotic dilution, with respect to the pure solvent compartment, [$N \cdot m^{-2}$].
ΔP_{cri}	The pressure caused by a column of liquid with the density of $\bar{\rho}_{\text{cri}}(\text{Test})$, [$N \cdot m^{-2}$].
π	The mathematical constant pi, [$3.14 \dots$].



ψ	The critical dilution factor, which causes the osmotic engine to a complete halt, [-].
π	The osmotic pressure of a binary solution, [$N \cdot m^{-2}$].
R	The universal gas constant, [$8.31 J \cdot K^{-1} \cdot mol^{-1}$].
$\rho(Test)$	The density of a <i>Test</i> solution, [$g \cdot cm^{-3}$].
$\tilde{\rho}(Test)$	The density of a <i>Test</i> solution after osmotic dilution in the working cylinder, [$g \cdot cm^{-3}$].
$\tilde{\rho}_{cri}(Test)$	The critical density of a <i>Test</i> solution after osmotic dilution in the working cylinder, [$g \cdot cm^{-3}$].
T	Kelvin temperature, [K].
$V(Test)$	The volume of a <i>Test</i> solution, [m^3].
V_C	The volume of a single cylinder in the osmotic engine, [m^3].
V_C^{eq}	The osmotic equilibrium volume, [m^3].
ΔV^{stop}	The increment of volume of a real binary <i>Test</i> solution with its own solvent resulting in an osmotic pressure less than or equal to the critical osmotic pressure, [m^3].
W	The PV -work done on the surroundings during osmotic dilution, [J].
$\partial W / \partial t$	The engine power measured on the crank, [$J \cdot s^{-1}$].
H_B	Henry's constant for the solute <i>B</i> , [$N \cdot m^{-2}$].
P_B^*	The vapour pressure of pure solute <i>B</i> , [$N \cdot m^{-2}$].
γ_B^R	The Raoult-based activity coefficient of solute <i>B</i> , [-].
γ_B^H	The Henry-based activity coefficient of solute <i>B</i> , [-].
a_B^R	The Raoult-based activity of solute <i>B</i> , [-].
a_B^H	The Henry-based activity of solute <i>B</i> , [-].



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View Article Online
DOI: 10.1039/D5CP03542B

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April 3 2025

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Data Availability Statement for the manuscript

“Measuring the Gibbs Free Energy of Mixing Using the Concept of an Osmotic Engine

All arguments and derivations supporting the findings of this study are included within the article. No additional data were generated or analyzed.

Sincerely yours,

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